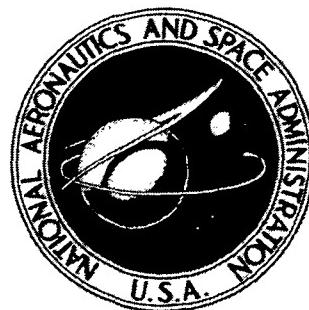


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COMPUTER PROGRAM FOR
COMPRESSIBLE LAMINAR OR TURBULENT
NONSIMILAR BOUNDARY LAYERS

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CONTENTS

| | Page |
|--|------|
| SUMMARY | 1 |
| INTRODUCTION | 1 |
| SYMBOLS | 2 |
| PROBLEM DISCUSSION | 7 |
| Basic Partial Differential Equations | 7 |
| Transformation to Conventional Similarity Coordinates | 10 |
| Numerical Computational Procedure | 14 |
| Eddy Viscosity and Turbulent Prandtl Number Formulations | 20 |
| Mixing-length relations and eddy-viscosity function for two-dimensional and axisymmetric flows | 20 |
| Eddy-viscosity function for flows on infinite swept cylinders | 22 |
| Turbulent Prandtl number formulations | 23 |
| PROGRAM DESCRIPTION | 24 |
| FORTRAN Variable List | 24 |
| Description, Flow Charts, and Listings of the Main Program and Subprograms | 38 |
| Main program D2630 | 38 |
| ABCDGS | 56 |
| COMPUTE | 58 |
| CALCM | 60 |
| USAGE | 65 |
| Input Description | 65 |
| Output Description | 69 |
| Sample Case | 71 |
| APPENDIX | 74 |
| REFERENCES | 79 |

COMPUTER PROGRAM FOR COMPRESSIBLE LAMINAR OR TURBULENT NONSIMILAR BOUNDARY LAYERS

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SUMMARY

A computer program is described which solves the two-dimensional and axisymmetric forms of the compressible-boundary-layer equations for continuity, mean momentum, and mean total enthalpy by an implicit finite-difference procedure. Turbulent flow is treated by the inclusion of an eddy viscosity model based upon a mixing-length formulation. The eddy conductivity is related to the eddy viscosity by the turbulent Prandtl number which may be an arbitrary function of the distance from the wall. The laminar-boundary-layer equations are recovered when the eddy viscosity is zero. Since a finite-difference procedure is used, the effects of variable wall and edge boundary conditions are easily included by modifying the program inputs.

INTRODUCTION

Several finite-difference methods are currently available for computing the development of compressible turbulent boundary layers (for example, refs. 1 to 6). The numerical procedures used in these methods are generally different, but results are similar when a common eddy viscosity formulation is employed. Therefore, the main difference between the various methods is in the formulation of the eddy viscosity and turbulent Prandtl number functions used to model the turbulence flux terms appearing in the mean-flow equations.

This report describes a computer program developed to solve the compressible-nonsimilar-boundary-layer equations for continuity, mean momentum, and total mean enthalpy for an ideal gas with constant specific heat. This program was used to obtain the results reported in references 7 and 8. An implicit finite-difference procedure similar to the procedures described in references 9 and 10 is used. The program will solve problems with the following flow configurations: (1) two-dimensional, (2) axisymmetric where the boundary-layer thickness is much less than the body radius, and (3) swept infinite cylinders.

The eddy viscosity is taken as a function of the local boundary-layer thickness, the normal distance from the wall, and the mean velocity gradient in the boundary layer. The formulation for the eddy viscosity is based on the mixing-length models of references 2 and 11. The turbulent Prandtl number may be either a constant or a specified tabulated function of the ratio of the normal distance from the wall to the boundary-layer thickness. By setting the eddy viscosity equal to zero, nonsimilar-laminar-boundary-layer flows can be computed. Since a finite-difference procedure is used, the effects of variable wall and edge boundary conditions and wall blowing or suction are easily included by modifying the program inputs.

The governing partial differential equations and their finite-difference forms are discussed first. Then, the digital computer program is described in detail including the flow charts, program code, and instructions for use with sample input and output.

SYMBOLS

A, B, C empirical constants in \bar{f}_{max} expression (see eq. (56a))

A^*, B^*, C^*, D^* coefficients in general difference equation for conservation of energy (see eq. (39))

$\hat{A}, \hat{B}, \hat{C}, \hat{D}$ coefficients in general difference equation for g-momentum (see eq. (38))

$\bar{A}, \bar{B}, \bar{C}, \bar{D}$ coefficients in general difference equation for F-momentum (see eq. (25))

A_b damping constant (see eq. (59))

A_d damping length (see eq. (59))

C_f skin-friction coefficient, $\frac{\tau_w}{\frac{1}{2} \rho_e u_e^2}$

c_p specific heat

F dimensionless chordwise velocity profile, u/u_e

\bar{f} mixing-length ratio, function of y/δ (see eq. (51))

\bar{G}, \bar{g} coefficients in formula for dependent variables (see eq. (31))

\hat{G}, \hat{g} coefficients in formula for dependent variables (see eq. (42))

| | |
|------------|--|
| G^*, g^* | coefficients in formula for dependent variables (see eq. (43)) |
| g | dimensionless spanwise velocity profile, w/w_e |
| H | total enthalpy, $h + \frac{u^2}{2} + \frac{w^2}{2}$ |
| H^* | form factor, δ^*/θ^* |
| h | static enthalpy |
| j | body-shape index ($j = 0$ for two-dimensional and swept-cylinder flows; $j = 1$ for axisymmetric flows) |
| K | ratio of successive $\Delta\eta$ steps (see eqs. (22) to (24)) |
| k | molecular conductivity |
| k^* | static eddy conductivity for turbulent flow, $-\overline{\frac{(\rho v)' h'}{\partial h / \partial y}}$ |
| L | reference length; a constant |
| l | mixing length |
| M | Mach number |
| \bar{M} | effective viscosity function for F-momentum equation (see eq. (10b)) |
| \hat{M} | effective viscosity function for g-momentum equation (see eq. (10c)) |
| M^*, M' | effective viscosity and turbulent Prandtl number functions for total mean enthalpy equation (see eqs. (10d) and (10e)) |
| m | computational grid index in x-direction (see fig. 2) |
| N | maximum value of n |
| N_{Pr} | molecular Prandtl number, $c_p \mu / k$ |
| $N_{Pr,T}$ | total turbulent Prandtl number, $c_p \epsilon / k$ |

| | |
|------------------|--|
| $N_{Pr,t}$ | static turbulent Prandtl number, $c_p \epsilon / k^*$ |
| N_{St} | Stanton number |
| n | computational grid index in y-direction (see fig. 2) |
| P | pressure gradient parameter (see eq. (10h)) |
| p | pressure |
| \bar{Q} | heat-transfer parameter, $\dot{q}_w L / (\mu_s H_e)$ |
| \dot{q} | heating rate |
| $R_{e,x}$ | Reynolds number based on x , $\frac{\rho_e u_e x}{\mu_e}$ |
| R_{e,δ^*} | Reynolds number based on displacement thickness, $\frac{\rho_e u_e \delta^*}{\mu_e}$ |
| R_{e,θ^*} | Reynolds number based on momentum thickness, $\frac{\rho_e u_e \theta^*}{\mu_e}$ |
| R_s | reference Reynolds number, $\frac{\rho_s \sqrt{2H_e} L}{\mu_s}$; a constant |
| r | dimensionless body radius (used only when $j = 1$), r_w / L |
| S | Sutherland's constant |
| T | temperature |
| U | velocity gradient parameter (see eq. (10g)) |
| u | mean physical velocity in streamwise (or chordwise for swept-cylinder problems) direction (see fig. 1) |
| v | transformed normal velocity (see eq. (11)) |
| v | mean physical velocity in direction normal to surface (see fig. 1) |
| w | mean physical velocity in spanwise direction (see fig. 1) |

| | |
|---------------|---|
| x | streamwise (or chordwise for swept-cylinder problems) direction in physical coordinate system (see fig. 1) |
| y | direction normal to surface in physical coordinate system (see fig. 1) |
| z | wall temperature gradient parameter (see eq. (10i)) |
| z | spanwise direction in physical coordinate system (see fig. 1) |
| δ | boundary-layer thickness in physical plane, taken where F or $g = 0.995$ |
| δ^* | displacement thickness, $\int_0^{y_N} \left(1 - \frac{\rho u}{\rho_e u_e}\right) dy$ |
| ϵ | eddy viscosity for turbulent flow, $-\frac{(\rho v)' u'}{\partial \bar{u} / \partial y}$ |
| ϵ_z | eddy viscosity in spanwise direction |
| ξ | dimensionless enthalpy ratio, H/H_e |
| η | transformed similarity coordinate (see eq. (8b)) |
| η_δ | value of η when $F = 0.995$ |
| θ | dimensionless enthalpy profile, $\frac{H - H_w}{H_e - H_w}$ |
| θ^* | momentum thickness, $\int_0^{y_N} \left(\frac{\rho u}{\rho_e u_e} \left(1 - \frac{u}{u_e}\right)\right) dy$ |
| κ | total eddy conductivity for turbulent flow, $-\frac{(\rho v)' H'}{\partial \bar{H} / \partial y}$ |
| Λ | sweep angle |
| μ | molecular viscosity |
| ξ | transformed similarity coordinate (see eq. (8a)) |
| $\tilde{\xi}$ | transformed streamwise length parameter (see eq. (10f)) |
| ρ | density |

| | |
|-------------|--|
| σ | error or convergence criteria (see eqs. (34) and (46) to (48)) |
| τ | shear stress |
| φ_r | density-viscosity product ratio (see eq. (10a)) |

Subscripts and superscripts:

| | |
|-----------|---|
| a | average value (see fig. 2 and eqs. (30a) and (30b)) |
| e | edge of boundary layer |
| F | F profile |
| g | g profile |
| i | incompressible value |
| j | body-shape index |
| max | maximum value |
| N | maximum value of n (see fig. 2 and eqs. (35), (44), and (45)) |
| n | index for points in y-direction |
| \bar{n} | similarity index (see eqs. (8b) and (9b)) |
| o | initial conditions at $x = x_0$ |
| r | variable reference value, evaluated at edge condition |
| res | resultant |
| s | constant reference value, usually taken as local isentropic stagnation conditions |
| t | isentropic stagnation value |
| w | wall |

z z-direction

∞ free stream

A prime on u , v , w , h , H , or ρ denotes a fluctuating quantity.

A bar over primed quantities indicates a time average.

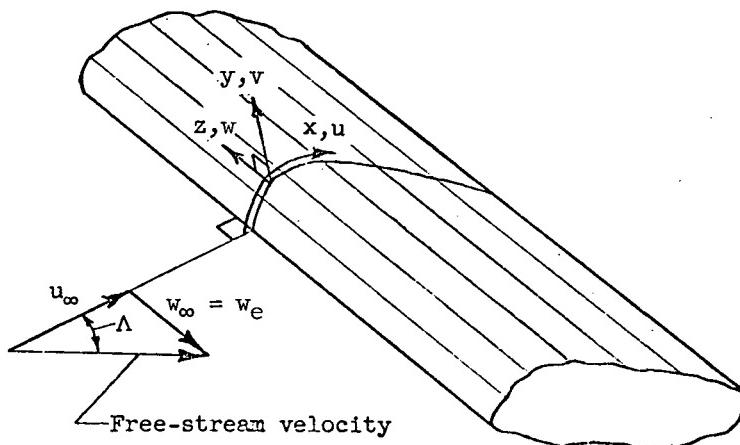


Figure 1.- Coordinate system for swept-leading-edge problem.

PROBLEM DISCUSSION

In this section, the partial differential equations and their finite-difference formulations are described. Also, details of the eddy-viscosity model are given. The basic physics of the problem is discussed, for example, in references 12 and 13.

Basic Partial Differential Equations

The partial differential equations in terms of physical coordinates and mean dimensional flow properties for boundary-layer flow are as follows (see refs. 12 and 13):

Continuity

$$\frac{\partial}{\partial x} (\rho u r^j) + \frac{\partial}{\partial y} (\rho v r^j) = 0 \quad (1)$$

where $j = 1$ for axisymmetric bodies and $j = 0$ for two-dimensional bodies and infinite swept cylinders.

x-momentum

$$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = - \frac{dp}{dx} + \frac{\partial}{\partial y} \left[\mu \left(1 + \frac{\epsilon}{\mu} \right) \frac{\partial u}{\partial y} \right] \quad (2)$$

where it is assumed that $p = p(x)$.

In order to compute the flow on an infinite swept cylinder (i.e., $\frac{\partial}{\partial z}() = 0$), it is necessary to incorporate the spanwise momentum equation. The spanwise coordinate is taken as z and the spanwise velocity as w ; hence,

z-momentum

$$\rho u \frac{\partial w}{\partial x} + \rho v \frac{\partial w}{\partial y} = \frac{\partial}{\partial y} \left[\mu \left(1 + \frac{\epsilon_z}{\mu} \right) \frac{\partial w}{\partial y} \right] \quad (3)$$

Note that $w = 0$ for all cases except when a swept-cylinder problem is specified.

Total energy

$$\rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left\{ \frac{\mu}{N_{Pr}} \left[\left(1 + \frac{\epsilon}{\mu} \frac{N_{Pr}}{N_{Pr,T}} \right) \frac{\partial H}{\partial y} - (1 - N_{Pr}) \left(u \frac{\partial u}{\partial y} + w \frac{\partial w}{\partial y} \right) \right] \right\} \quad (4)$$

The eddy viscosity ϵ and turbulent Prandtl number $N_{Pr,T}$ are supplied as simple functions of the local flow properties (within the boundary layer and/or in the free stream) and the distance from the wall. All flow quantities shown are time mean values. For an axisymmetric body, r is the dimensionless local radius of the body ($r = r_w/L$), and the assumption is made that the boundary-layer thickness δ is much less than r_w .

Solutions to equations (1) to (4) are sought in terms of u , v , w , and H . Auxiliary equations for the other variables are shown subsequently. Besides initial conditions (at $x = x_0$), the boundary conditions are:

y = 0 (wall or surface of body)

$$\left. \begin{array}{l} u = w = 0 \\ v = v_w(x) \\ H = H_w(x) \end{array} \right\} \quad (5a)$$

(Note that the adiabatic-wall boundary condition is not included in the program described herein and only air-to-air blowing can be considered.)

$$\left. \begin{array}{l} u \rightarrow u_e(x) \\ w \rightarrow w_e \text{ (a constant)} \\ H \rightarrow H_e \text{ (a constant)} \end{array} \right\} \quad (5b)$$

For convenience, dimensionless profile functions are introduced:

$$\left. \begin{aligned} F &= \frac{u}{u_e} \\ g &= \frac{w}{w_e} \\ \theta &= \frac{H - H_w}{H_e - H_w} \equiv \frac{\zeta - \zeta_w}{1 - \zeta_w} \end{aligned} \right\} \quad (6)$$

where $u_e = u_e(x)$, $\zeta_w = \zeta_w(x)$, and w_e and H_e are constants. Because of conditions (5a) and (5b), the boundary conditions for these new variables are

y = 0

$$F_w = g_w = \theta_w = 0 \quad (7a)$$

y → ∞

$$F_e = g_e = \theta_e = 1 \quad (7b)$$

Equations (1) to (4) may then be written as

Continuity

$$\frac{\partial}{\partial x} (\rho u_e F r^j) + \frac{\partial}{\partial y} (\rho v r^j) = 0 \quad (1a)$$

F-momentum

$$\rho u \frac{\partial F}{\partial x} + \rho v \frac{\partial F}{\partial y} = -\rho F^2 \frac{du_e}{dx} - \frac{1}{u_e} \frac{dp}{dx} + \frac{\partial}{\partial y} \left[\mu \left(1 + \frac{\epsilon}{\mu} \right) \frac{\partial F}{\partial y} \right] \quad (2a)$$

g-momentum

$$\rho u \frac{\partial g}{\partial x} + \rho v \frac{\partial g}{\partial y} = \frac{\partial}{\partial y} \left[\mu \left(1 + \frac{\epsilon z}{\mu} \right) \frac{\partial g}{\partial y} \right] \quad (3a)$$

Total energy

$$\begin{aligned} \rho u \frac{\partial \theta}{\partial x} + \rho v \frac{\partial \theta}{\partial y} &= -\rho u \frac{1 - \theta}{1 - \zeta_w} \frac{d\zeta_w}{dx} + \frac{\partial}{\partial y} \left\{ \frac{\mu}{N_{Pr}} \left[\left(1 + \frac{\epsilon}{\mu} \frac{N_{Pr}}{N_{Pr,T}} \right) \frac{\partial \theta}{\partial y} \right. \right. \\ &\quad \left. \left. - \frac{1 - N_{Pr}}{1 - \zeta_w} \left(\frac{u_e^2}{2H_e} \frac{\partial F^2}{\partial y} + \frac{w_e^2}{2H_e} \frac{\partial g^2}{\partial y} \right) \right] \right\} \end{aligned} \quad (4a)$$

Transformation to Conventional Similarity Coordinates

A finite-difference computing procedure will be used to solve equations (1a) to (4a). In order to keep the number of steps across the boundary layer (i.e., in the y -direction) approximately constant and also to take advantage of similar profile shapes which exist for certain conditions, a similarity transformation of x and y is introduced as follows:

$$\xi\left(\frac{x}{L}\right) = R_s \int_0^{x/L} \frac{(\rho\mu)_e}{(\rho\mu)_s} \frac{u_e}{\sqrt{2H_e}} r^{2j} d \frac{x}{L} \quad (8a)$$

$$\eta\left(\frac{x}{L}, \frac{y}{L}\right) = R_s \frac{u_e / \sqrt{2H_e}}{(2\xi)^{\bar{n}}} r^j \int_0^{y/L} \frac{\rho}{\rho_s} d \frac{y}{L} \quad (8b)$$

The quantity \bar{n} is taken as a constant, generally equal to about 0.8 for turbulent flows and 0.5 for laminar flows.

Since $\partial\xi/\partial y = 0$, the general transformation formulas from equations (8a) and (8b) are

$$\frac{\partial}{\partial x} = \frac{(\rho\mu)_e u_e}{\mu_s^2} r^{2j} \frac{\partial}{\partial \xi} + \frac{\partial \eta}{\partial x} \frac{\partial}{\partial \eta} \quad (9a)$$

$$\frac{\partial}{\partial y} = \frac{\rho u_e}{\mu_s} \frac{r^j}{(2\xi)^{\bar{n}}} \frac{\partial}{\partial \eta} \quad (9b)$$

For the sake of brevity, several special definitions are introduced:

$$\varphi_r = \frac{\rho\mu}{(\rho\mu)_e} \quad (10a)$$

$$\bar{M} = \varphi_r \left(1 + \frac{\epsilon}{\mu}\right) \quad (10b)$$

$$\hat{M} = \varphi_r \left(1 + \frac{\epsilon_z}{\mu}\right) \quad (10c)$$

$$M^* = \frac{\varphi_r}{N_{Pr}} \left(1 + \frac{\epsilon}{\mu} \frac{N_{Pr}}{N_{Pr,T}}\right) \quad (10d)$$

$$M' = \frac{\varphi_r}{N_{Pr}} \frac{1 - N_{Pr}}{1 - \xi_w} \quad (10e)$$

$$\bar{\xi} = (2\xi)^{2\bar{n}} \quad (10f)$$

$$U = \frac{1}{u_e/\sqrt{2H_e}} \frac{d(u_e/\sqrt{2H_e})}{d\xi} \quad (10g)$$

$$P = \frac{1}{\rho_e u_e^2} \frac{dp}{d\xi} \quad (10h)$$

$$Z = \frac{1}{1 - \zeta_w} \frac{d\xi_w}{d\xi} \quad (10i)$$

In the present computer program $P = -U$, which requires constant entropy flow external to the boundary layer. When the entropy is not constant, $P \neq -U$ and the external flow conditions would be determined from an iteration to balance the mass flow in the boundary layer with that of the upstream inviscid flow. (See ref. 14.)

The transformed normal velocity is defined as

$$V = \frac{\mu_s^{2\xi}}{(\rho\mu)_e u_e r^{2j}} \left[F \frac{\partial\eta}{\partial x} + \frac{\rho v r^j}{\mu_s (2\xi)^{\bar{n}}} \right] \quad (11)$$

Application of equations (9a) and (9b) to equations (1a) to (4a) then results in (see ref. 9)

Continuity

$$\bar{\xi} \frac{\partial F}{\partial \xi} + \frac{\partial V}{\partial \eta} + \bar{\xi} F \frac{\bar{n}}{\xi} = 0 \quad (12)$$

F-momentum

$$\bar{\xi} F \frac{\partial F}{\partial \xi} + V \frac{\partial F}{\partial \eta} = -\bar{\xi} \left(U F^2 + P \frac{\rho_e}{\rho} \right) + \frac{\partial}{\partial \eta} \left(\bar{M} \frac{\partial F}{\partial \eta} \right) \quad (13)$$

g-momentum

$$\bar{\xi} F \frac{\partial g}{\partial \xi} + V \frac{\partial g}{\partial \eta} = \frac{\partial}{\partial \eta} \left(\hat{M} \frac{\partial g}{\partial \eta} \right) \quad (14)$$

Total energy

$$\bar{\xi} F \frac{\partial \theta}{\partial \xi} + V \frac{\partial \theta}{\partial \eta} = -\bar{\xi} Z F (1 - \theta) + \frac{\partial}{\partial \eta} \left[M^* \frac{\partial \theta}{\partial \eta} - M' \left(\frac{u_e^2}{2H_e} \frac{\partial F^2}{\partial \eta} + \frac{w_e^2}{2H_e} \frac{\partial g^2}{\partial \eta} \right) \right] \quad (15)$$

The boundary conditions on the profile functions (eqs. (7a) and (7b)) are restated as follows:

$$\underline{\eta = 0}$$

$$\left. \begin{aligned} F_w &= g_w = \theta_w = 0 \\ v_w &= \begin{cases} 0 & \text{or} \\ v_w(x), \text{ specified function} \end{cases} \end{aligned} \right\} \quad (16a)$$

$$\underline{\eta = \eta_e}$$

$$F_e = g_e = \theta_e = 1.0 \pm \text{Error criteria} \quad (16b)$$

To make the system given by equations (12) to (15) determinate, several auxiliary functions are required. The following are given functions of x :

$$\left. \begin{aligned} \frac{u_e}{\sqrt{2H_e}} &= \frac{u_e}{\sqrt{2H_e}}(x) \\ \frac{\rho_e}{\rho_s} &= \frac{\rho_e}{\rho_s}(x) \\ r &= r(x) \\ \zeta_w &= \zeta_w(x) \\ \frac{p_e}{p_s} &= \frac{p_e}{p_s}(x) \end{aligned} \right\} \quad (17a)$$

To avoid problems in numerical differentiation for sparse input data, the following derivatives can also be specified:

$$\left. \begin{aligned} \frac{d(u_e/\sqrt{2H_e})}{d(\frac{x}{L})} &= \frac{d(u_e/\sqrt{2H_e})(x)}{d(\frac{x}{L})} \\ \frac{d\zeta_w}{d(\frac{x}{L})} &= \frac{d\zeta_w}{d(\frac{x}{L})}(x) \end{aligned} \right\} \quad (17b)$$

For a perfect gas with constant specific heats, the static enthalpy ratio, computed from profile functions, is

$$\frac{h}{h_e} \equiv \frac{\rho_e}{\rho} = \frac{(1 - \xi_w)\theta + \xi_w - \frac{u_e^2}{2H_e} F^2 - \frac{w_e^2}{2H_e} g^2}{1 - \frac{u_e^2}{2H_e} - \frac{w_e^2}{2H_e}} \quad (18)$$

Sutherland's viscosity relation is used in the density-viscosity product ratio (eq. (10a)) to give

$$\varphi_r = \left(\frac{h/H_e}{h_e/H_e} \right)^{1/2} \frac{\frac{h_e}{H_e} + \frac{S}{H_e}}{\frac{h}{H_e} + \frac{S}{H_e}} \quad (19)$$

which can be utilized for various diatomic gases. The value of Sutherland's constant S would depend on the gas and the temperature range of the problem. The enthalpy ratios in equation (19) are

$$\frac{h}{h_e} = (1 - \xi_w)\theta + \xi_w - \frac{u_e^2}{2H_e} F^2 - \frac{w_e^2}{2H_e} g^2 \quad (20)$$

and

$$\frac{h_e}{H_e} = 1 - \frac{u_e^2}{2H_e} - \frac{w_e^2}{2H_e} \quad (21)$$

where

N_{Pr} = Constant

$N_{Pr,T}$ either a constant or a specified function of y/δ

$\frac{\epsilon}{\mu}$ function of profile parameters and y/δ

δ value of y where F or $g = 0.995$ (note that this is not the asymptotic boundary condition on the computed profiles)

$\xi = \xi\left(\frac{x}{L}\right)$ (see eq. (8a))

Numerical Computational Procedure

The system of equations (12) to (15) is solved by an implicit finite-difference procedure. The grid to be used and the node notation is illustrated in figure 2. To increase the accuracy and efficiency, a variable grid size in the η - and ξ -directions may be used. The initial distribution of $\Delta\eta$ will generally be held fixed for a given problem.

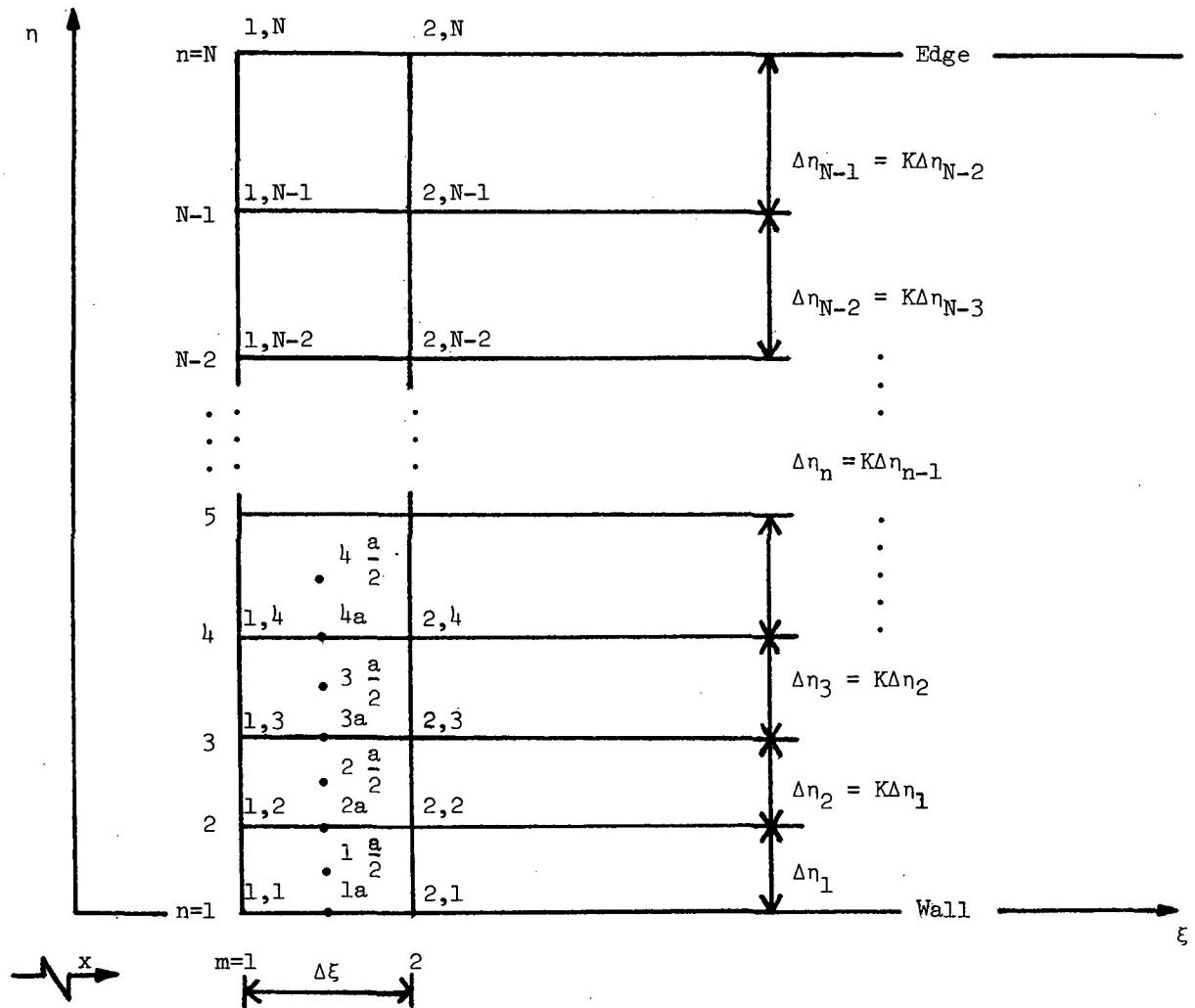


Figure 2.- Grid notation.

As illustrated in figure 2, the letter "a" indicates the average value of a quantity between the two x-locations under consideration and the notation "a/2" indicates the average value of a quantity at the four adjacent node points.

In the present procedure, the solution is advanced downstream in the x- or ξ -direction by redefining the $m = 2$ values as $m = 1$ values and a new $m = 2$ station is chosen a distance $\Delta\xi$ downstream; that is, a two-point difference scheme in the x- or ξ -direction is used.

It can be seen that if K is a constant, the successive values of $\Delta\eta$ form a geometric progression. Hence

$$\Delta\eta_n = K^{n-1} \Delta\eta_1 \quad (22)$$

Since the total number of $\Delta\eta$ steps across the boundary layer is $N - 1$, the size of the last step at $n = N - 1$ is

$$\Delta\eta_{N-1} = K^{N-2} \Delta\eta_1 \quad (23)$$

The thickness of the boundary layer η_e is given by

$$\eta_e = \Delta\eta_1 \frac{1 - K^{N-1}}{1 - K} \quad (24)$$

Thus, if η_e , $\Delta\eta_1$, and the number of steps ($N - 1$) are specified, K and $\Delta\eta_{N-1}$ can be determined. Note that if $\Delta\eta = \text{Constant}$ is desired, $K = 1.0$. Generally, the value of K will be a constant slightly greater than 1.0. (The value $K = 1.02$ is usually used.)

The input is specified at station $m = 1$ from $n = 1$ to $n = N$ from which values of all variables are to be computed at the next station ($m = 2$). Equation (12) with $\frac{\partial F}{\partial \xi} = 0$ is used to obtain initial values of V . The various derivatives in equations (12) to (15) are replaced by linear difference quotients (see refs. 9 and 10) and the equations are evaluated at the intermediate station. Consider first the F-momentum equation (eq. (13)). At point 3a, the difference equation becomes

$$\begin{aligned} & \bar{\xi}_a F_{3a} \frac{F_{2,3} - F_{1,3}}{\Delta\xi} + \frac{V_{3,a}}{2} \frac{F_{1,4} + F_{2,4} - F_{1,2} - F_{2,2}}{\Delta\eta_2 + \Delta\eta_3} \\ &= -\bar{\xi}_a \left[U_a F_{3a}^2 + P_a \left(\frac{\rho_e}{\rho} \right)_{3a} \right] + \frac{\bar{M}_{3a} \frac{F_{2,4} + F_{1,4} - F_{1,3} - F_{2,3}}{2\Delta\eta_3} - \bar{M}_{2a} \frac{F_{2,3} + F_{1,3} - F_{2,2} - F_{1,2}}{2\Delta\eta_2}}{\frac{1}{2}(\Delta\eta_3 + \Delta\eta_2)} \end{aligned}$$

The coefficients of the unknown quantities $F_{2,4}$, $F_{2,3}$, and $F_{2,2}$ are collected and equation (13) evaluated at point 3a may be written as

$$\bar{A}_3 F_{2,4} + \bar{B}_3 F_{2,3} + \bar{C}_3 F_{2,2} = \bar{D}_3$$

The general form of this equation is then

$$\bar{A}_n F_{2,n+1} + \bar{B}_n F_{2,n} + \bar{C}_n F_{2,n-1} = \bar{D}_n \quad (25)$$

where

$$\bar{A}_n = \frac{V_{na}}{2(\Delta\eta_n + \Delta\eta_{n-1})} \frac{\bar{M}_{n\frac{a}{2}} \left(\frac{2}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \quad (26)$$

$$\bar{B}_n = \frac{\bar{\xi}a}{\Delta\xi} F_{na} + \frac{\bar{M}_{n\frac{a}{2}} \left(\frac{2}{1+K} \right) + \bar{M}_{(n-1)\frac{a}{2}} \left(\frac{2K}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \quad (27)$$

$$\bar{C}_n = - \left[\frac{V_{na}}{2(\Delta\eta_n + \Delta\eta_{n-1})} + \frac{\bar{M}_{n\frac{a}{2}} \left(\frac{2K}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \right] \quad (28)$$

$$\begin{aligned} \bar{D}_n = & -\bar{A}_n F_{1,n+1} + \left[\frac{\bar{\xi}a}{\Delta\xi} F_{na} - \frac{\bar{M}_{n\frac{a}{2}} \left(\frac{2}{1+K} \right) + \bar{M}_{(n-1)\frac{a}{2}} \left(\frac{2K}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \right] F_{1,n} \\ & - \bar{C}_n F_{1,n-1} - (\bar{\xi}U)_a F_{na}^2 - (\bar{\xi}P)_a \left(\frac{\rho_e}{\rho} \right)_{na} \end{aligned} \quad (29)$$

In equations (26) to (29), any function that depends only on ξ is denoted by subscript a , which indicates the average value of the function between stations 1 and 2. Conditions at stations 1 and 2 are updated as the solution proceeds in the x -direction by a general iteration procedure. For the first iteration in this procedure, the "a" or average values of quantities in the \bar{A}_n , \bar{B}_n , \bar{C}_n , and \bar{D}_n functions are evaluated at the $m = 1$ station. On successive iterations, the "a" values are updated using the latest values from the $m = 2$ station; that is, the "a" values are taken as simple numerical averages of the appropriate node points. Examples of these are as follows:

$$\left. \begin{aligned} F_{na} &= \frac{1}{2}(F_{1,n} + F_{2,n}) \\ F_{\frac{n}{2}a} &= \frac{1}{2}[F_{(n+1)a} + F_{na}] \end{aligned} \right\} \quad (30a)$$

Hence,

$$F_{\frac{n}{2}a} = \frac{1}{4}(F_{1,n+1} + F_{2,n+1} + F_{1,n} + F_{2,n}) \quad (30b)$$

Since boundary conditions are specified at $n = 1$ and $n = N$, equation (25) represents a system of $N - 2$ equations with $N - 2$ unknowns (the values of $F_{2,n}$ from $n = 2$ to $n = N - 1$). Since the matrix of the system given by equation (25) is tridiagonal, the unknown F_2 values are easily obtained by successive elimination of unknowns with the formula (see ref. 15)

$$F_{2,n} = \bar{G}_n F_{2,n+1} + \bar{g}_n \quad (31)$$

which is applied by starting at the outer boundary ($n = N$) where $F_e \approx 1.0$ and proceeding down to the wall ($n = 1$) where $F = 0$. The functions \bar{G}_n and \bar{g}_n are computed from the following recursion formulas:

$$\left. \begin{aligned} \bar{G}_n &= \frac{-\bar{A}_n}{\bar{B}_n + \bar{C}_n \bar{G}_{n-1}} \\ \bar{g}_n &= \frac{\bar{D}_n - \bar{C}_n \bar{g}_{n-1}}{\bar{B}_n + \bar{C}_n \bar{G}_{n-1}} \end{aligned} \right\} \quad (32)$$

From the wall boundary condition ($F_{2,1} = 0$) applied to equation (31), $\bar{G}_1 = \bar{g}_1 = 0$. Therefore, from the recursion relations (32),

$$\left. \begin{aligned} \bar{G}_2 &= \frac{-\bar{A}_2}{\bar{B}_2} \\ \bar{g}_2 &= \frac{\bar{D}_2}{\bar{B}_2} \end{aligned} \right\} \quad (33)$$

The functions \bar{G}_n and \bar{g}_n are computed from equations (32) by starting at the wall (actually at $n = 2$) and working out to the outer boundary ($n = N - 1$). Equation (31) then supplies the required values of $F_{2,n}$ by starting at the outer edge ($n = N - 1$) and proceeding down to the wall ($n = 2$). Before this procedure can be completed, it is necessary to know the number of equations (or $\Delta\eta$ steps) to be used. The correct number of steps is determined from the physical requirement that

$$\left(\frac{\partial F}{\partial \eta}\right)_{\eta=\eta_e} \leq \sigma_e \quad (34)$$

where σ_e is some small specified error criterion. Equation (34) is written in finite-difference form as

$$F_{2,N} - F_{2,N-1} \leq \Delta \eta_{N-1} \sigma_e$$

From equation (31),

$$F_{N-1} = \bar{G}_{N-1} F_N + \bar{g}_{N-1}$$

Then, since $F_N = F_e \approx 1.0$, the values of \bar{G}_{N-1} and \bar{g}_{N-1} must satisfy the inequality

$$\left| F_e \left(1 - \bar{G}_{N-1} \right) - \bar{g}_{N-1} \right| \leq \left| \Delta \eta_{N-1} \sigma_e \right| \quad (35)$$

where F_e and σ_e are specified and $\Delta \eta_{N-1}$ is computed from equation (23). Hence the value of $n = N - 1$ is obtained by inserting successive values of \bar{G}_n and \bar{g}_n from equations (32) (as the suspected neighborhood of $n \approx N - 10$, say, is approached) into inequality (35).

After the set of $F_{2,n}$ values is obtained, the values of V_{na} are updated by using equation (12), which is written in the appropriate finite-difference form evaluated at $\frac{n^a}{2}$ (average of four adjacent node points). For example, at $\frac{2^a}{2}$ there is obtained

$$V_{3a} = V_{2a} - \frac{\Delta \eta_2 \bar{\xi}_a}{2 \Delta \xi} (F_{2,3} + F_{2,2} - F_{1,3} - F_{1,2}) - \Delta \eta_2 \bar{\xi}_a \frac{\bar{n}}{\bar{\xi}_a} F_{2, \frac{a}{2}} \quad (36)$$

As noted previously, input values of V_{na} for the first iteration at the input ξ station are computed from equation (12) by dropping derivatives with respect to ξ and evaluating at $m = 1$. The result is

$$V_{3a} = V_{2a} - \frac{1}{2} \Delta \eta_2 \bar{\xi}_1 (F_{1,3} + F_{1,2}) \frac{\bar{n}}{\bar{\xi}_1} \quad (37)$$

General expressions for equations (36) and (37) follow from inspection. The values of V_{na} are computed by starting at the known boundary condition V_{1a} (that is, $V_{w,a}$) and applying the general form of equation (36) (or eq. (37) for the first iteration at the input station) successively out to $n = N$.

With these improved values of V_{na} from equation (36), the first approximations of $g_{2,n}$ and $\theta_{2,n}$ are computed from equations (14) and (15). These equations are written in finite-difference form by a procedure identical to that illustrated for the F -momentum equation. The results for equations (14) and (15) are

$$\hat{A}_n g_{2,n+1} + \hat{B}_n g_{2,n} + \hat{C}_n g_{2,n-1} = \hat{D}_n \quad (38)$$

$$A_n^* \theta_{2,n+1} + B_n^* \theta_{2,n} + C_n^* \theta_{2,n-1} = D_n^* \quad (39)$$

The expressions for $\hat{A}_n, \hat{B}_n, \hat{C}_n$ and A_n^*, B_n^*, C_n^* are identical to those for equation (13), as given by equations (26), (27), and (28), except that the \bar{M} values are replaced by \hat{M} and M^* , respectively. The "D" quantities are computed from the following equations:

$$\hat{D}_n = -\hat{A}_n g_{1,n+1} + \left[\frac{\xi a}{\Delta \xi} F_{na} - \frac{\hat{M}_{n/2} \left(\frac{2}{1+K} \right) + \hat{M}_{(n-1)/2} \left(\frac{2K}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \right] g_{1,n} - \hat{C}_n g_{1,n-1} \quad (40)$$

$$\begin{aligned} D_n^* &= -A_n^* \theta_{1,n+1} + \left[\frac{\xi a}{\Delta \xi} F_{na} - \frac{M_{n/2}^* \left(\frac{2}{1+K} \right) + M_{(n-1)/2}^* \left(\frac{2K}{1+K} \right)}{2\Delta\eta_n \Delta\eta_{n-1}} \right] \theta_{1,n} - C_n^* \theta_{1,n-1} \\ &\quad + (\xi Z)_a F_{na} (\theta_{na} - 1) - \frac{(u_e^2)_a}{2H_e 2\Delta\eta_n \Delta\eta_{n-1}} \left[M'_{n/2} \left(\frac{2}{1+K} \right) (F_{1,n+1}^2 + F_{2,n+1}^2 - F_{1,n}^2 - F_{2,n}^2) \right. \\ &\quad \left. - M'_{(n-1)/2} \left(\frac{2K}{1+K} \right) (F_{1,n}^2 + F_{2,n}^2 - F_{1,n-1}^2 - F_{2,n-1}^2) \right] \\ &\quad - \frac{w_e^2}{2H_e 2\Delta\eta_n \Delta\eta_{n-1}} \left[M'_{n/2} \left(\frac{2}{1+K} \right) (g_{1,n+1}^2 + g_{2,n+1}^2 - g_{1,n}^2 - g_{2,n}^2) \right. \\ &\quad \left. - M'_{(n-1)/2} \left(\frac{2K}{1+K} \right) (g_{1,n}^2 + g_{2,n}^2 - g_{1,n-1}^2 - g_{2,n-1}^2) \right] \end{aligned} \quad (41)$$

The formulas for g_2 and θ_2 are of the same form as equation (31) (since the wall boundary conditions for g and θ are the same as those for F) and are written as

$$g_{2,n} = \hat{G}_n g_{2,n+1} + \hat{g}_n \quad (42)$$

$$\theta_{2,n} = G_n^* \theta_{2,n+1} + g_n^* \quad (43)$$

where the recursion formulas for the \bar{G} and \tilde{g} functions are of the same form as equation (32) with the appropriate superscript notation supplied. The number of equations in the systems given by equations (42) and (43) is again determined from

$$\left| g_e \left(1 - \hat{G}_{N-1} \right) - \hat{g}_{N-1} \right| \leq \left| \Delta \eta_{N-1} \sigma_e \right| \quad (44)$$

$$\left| \theta_e \left(1 - G_{N-1}^* \right) - g_{N-1}^* \right| \leq \left| \Delta \eta_{N-1} \sigma_e \right| \quad (45)$$

where, as before, g_e , σ_e , and θ_e are specified. An iteration procedure is thus used where equations (25), (36), (38), and (39) are solved, in that order. Equation (37) is used only on the first iteration. When the following convergence criteria are satisfied, the iterations are stopped and the entire procedure is repeated at the next ξ step. The convergence criteria are written as

$$\frac{F_{2,2,i+1} - F_{2,2,i}}{F_{2,2,i}} \leq \sigma_w \quad (46)$$

$$\frac{g_{2,2,i+1} - g_{2,2,i}}{g_{2,2,i}} \leq \sigma_w \quad (47)$$

$$\frac{\theta_{2,2,i+1} - \theta_{2,2,i}}{\theta_{2,2,i}} \leq \sigma_w \quad (48)$$

where the index i denotes the number of the iteration cycle at a given ξ station.

Eddy Viscosity and Turbulent Prandtl Number Formulations

In this section, the models used for the turbulence correlation terms appearing in the conservation equations for the mean flow are described. The eddy-viscosity formulations are discussed first and then the turbulent Prandtl number expressions are discussed. For a more detailed discussion of these models, see reference 7.

Mixing-length relations and eddy-viscosity function for two-dimensional and axisymmetric flows. - The turbulent shear for two-dimensional flow is given by

$$\tau_t = \epsilon \frac{\partial u}{\partial y} \quad (49)$$

A mixing length l may be defined by the relation (see refs. 2 and 11)

$$\tau_t = \rho l^2 \left| \frac{\partial u}{\partial y} \right| \frac{\partial u}{\partial y} \quad (50)$$

From the calculations in reference 11 (based on experimental data), it was shown that l/δ tends to be a nearly universal function of y/δ and, in the outer portion of the boundary layer, l/δ is approximately constant and equal to a typical incompressible value of 0.09 (at least for adiabatic flows up to $M_e = 5.0$). Hence, it is assumed herein that

$$\frac{l}{\delta} = \bar{f}\left(\frac{y}{\delta}\right) \quad (51)$$

where

$$\frac{y}{\delta} = \frac{\int_0^{\eta} \frac{\rho_e}{\rho} d\eta}{\int_0^{\eta_\delta} \frac{\rho_e}{\rho} d\eta} \quad (52)$$

The eddy viscosity is then given by the equation

$$\epsilon = \rho \delta^2 \left(\frac{l}{\delta} \right)^2 \left| \frac{\partial u}{\partial y} \right| \quad (53)$$

Transformation to the η coordinate then gives

$$\frac{\epsilon}{\mu} = (2\xi)^{\bar{n}} \left(\frac{l}{\delta} \right)^2 \left| \frac{\partial F}{\partial \eta} \right| \frac{\mu_s(\rho)}{\mu} \left(\frac{\rho}{\rho_e} \right)^2 r^{-j} \left(\int_0^{\eta_\delta} \frac{\rho_e}{\rho} d\eta \right)^2 \quad (54)$$

where η_δ is the value of η when $F = 0.995$. The appropriate finite-difference form for equation (54) is illustrated by evaluating the equation at point (1,n) as follows:

$$\left(\frac{\epsilon}{\mu} \right)_{1,n} = \left[\frac{(2\xi)^{\bar{n}}}{r^j} \right]_1 \left[\left(\frac{l}{\delta} \right)^2 \frac{\mu_s(\rho)}{\mu} \left(\frac{\rho}{\rho_e} \right)^2 \left(\int_0^{\eta_\delta} \frac{\rho_e}{\rho} d\eta \right)^2 \right]_{1,n} \left| \frac{F_{1,n+1} - F_{1,n-1}}{\Delta \eta_{n-1} + \Delta \eta_n} \right| \quad (55)$$

Note that the quantity ϵ/μ enters the viscosity functions, which are always evaluated at either $n \frac{a}{2}$ or $(n-1) \frac{a}{2}$ as illustrated for \bar{M} by equations (26) to (29). These values of $\bar{M}_{n \frac{a}{2}}$ or $\bar{M}_{(n-1) \frac{a}{2}}$ are calculated from the arithmetical average of the four surrounding mesh points as illustrated by equation (30b). The \bar{f} function is either a tabulated function of y/δ or it may be computed from the following expressions:

$$\left. \begin{aligned} \bar{f} &= 0.4 \frac{y}{\delta} & \left(\frac{y}{\delta} \leq 0.1 \right) \\ \bar{f} &= 0.04 + \frac{y - 0.1}{0.2} (\bar{f}_{max} - 0.04) & \left(0.1 \leq \frac{y}{\delta} \leq 0.3 \right) \\ \bar{f} &= \bar{f}_{max} & \left(\frac{y}{\delta} \geq 0.3 \right) \end{aligned} \right\} \quad (56)$$

where

$$\bar{f}_{\max} = A + BH_i^* + CH_i^{*2} \quad (56a)$$

and

$$H_i^* \equiv \frac{\int_0^\delta (1 - F)dy}{\int_0^\delta F(1 - F)dy} \quad (57)$$

Recommended values of A , B , and C are given in reference 7.

In the wall region, the basic mixing-length function (eqs. (56)) is modified to account for the turbulent damping effect of the wall by multiplying \bar{f} by the Van Driest exponential damping function (see ref. 7). The final mixing length then becomes

$$\frac{l}{\delta} = \left[1 - \exp \left(- \frac{y/L}{A_d/L} \right) \right] \bar{f} \quad (58)$$

where, if A_d is based on wall properties,

$$A_d = \frac{A_b \mu_w}{\rho_w \sqrt{\tau_w / \rho_w}}$$

or

$$\left(\frac{A_d}{L} \right)_m = \frac{A_b (2\xi)^{n/2}}{R_s \left(\frac{\mu_s}{\mu_w} \right)^{1/2} \frac{\rho_w}{\rho_e} \frac{\rho_e}{\rho_s} \frac{u_e}{\sqrt{2H_e}} r^{j/2} \left(\frac{F_m,2}{\Delta \eta_1} \right)^{1/2}} \quad (59)$$

For $v_w = 0$, $A_b = 26$; and for $v_w \neq 0$ (blowing or suction), the A_b value is obtained from an input table of A_b values as a function of $\frac{\rho_w v_w}{\rho_e u_e} \frac{2}{C_f}$. (See ref. 7.) Computa-

tions can also be carried out by using the program with $A_d = \frac{A_b \mu}{\rho \sqrt{\tau_w / \rho}}$; that is, A_d can be based on wall shear and local properties in the boundary layer.

Eddy-viscosity function for flows on infinite swept cylinders. - An eddy-viscosity formulation suitable for flows on infinite swept cylinders was derived in reference 8. This formulation is similar to that for two-dimensional and axisymmetric flows except that $\left| \frac{\partial u}{\partial y} \right|$ is replaced by $\sqrt{\left(\frac{\partial u}{\partial y} \right)^2 + \left(\frac{\partial w}{\partial y} \right)^2}$ in equation (53). The eddy-viscosity relation evaluated at the point $1,n$ then becomes

$$\left(\frac{\epsilon}{\mu}\right)_{1,n} = \left[\frac{(2\xi)^n}{r^j} \right]_1 \left[\left(\frac{l}{\delta} \right)^2 \frac{\mu_s}{\mu} \left(\frac{\rho}{\rho_e} \right)^2 \left(\int_0^{\eta_n} \frac{\rho_e}{\rho} d\eta \right)^2 \right]_{1,n} \left[\left(\frac{F_{1,n+1} - F_{1,n-1}}{\Delta\eta_{n-1} + \Delta\eta_n} \right)^2 + \left(\frac{g_{1,n+1} - g_{1,n-1}}{\Delta\eta_{n-1} + \Delta\eta_n} \right)^2 \left(\frac{w_e / \sqrt{2H_e}}{u_e / \sqrt{2H_e}} \right)^2 \right]^{1/2} \quad (60)$$

This formulation for ϵ/μ is used in both the chordwise and spanwise momentum equations (eqs. (13) and (14)); that is, $\epsilon_z = \epsilon$. The wall damping function (eq. (58)) is applied to the swept-cylinder problem by assuming that $\sqrt{\tau_w/\rho_w}$ is based on the total shear at the wall. This total or resultant shear is the vector sum of the shear in the chordwise and spanwise directions. Equation (59) then becomes

$$\left(\frac{A_d}{L} \right)_m = \frac{A_b (2\xi)^{n/2}}{R_s \left(\frac{\mu_s}{\mu_w} \right)^{1/2} \frac{\rho_w}{\rho_e} \frac{\rho_e}{\rho_s} \frac{u_e}{2H_e} r^{j/2} \left[\left(\frac{F_{m,2}}{\Delta\eta_1} \right)^2 + \left(\frac{g_{m,2}}{\Delta\eta_1} \right)^2 \left(\frac{w_e / \sqrt{2H_e}}{u_e / \sqrt{2H_e}} \right)^2 \right]^{1/4}} \quad (61)$$

Also, for $v_w \neq 0$, the A_b value is taken as a function of $\frac{\rho_w v_w}{\rho_e (u_e^2 + w_e^2)^{1/2}} \frac{2}{C_f, \text{res}}$. (See ref. 8.)

Turbulent Prandtl number formulations. - The term involving eddy conductivity which appears in the conservation equation for total energy (eq. (4a)) has been modeled in the program by using two approaches. The first of these involves modeling the term

$$\overline{-(\rho v)' H'} = \kappa \frac{\partial \bar{H}}{\partial y} \quad (62)$$

which results in the definition of a "total turbulent Prandtl number"

$$N_{Pr,T} \equiv \frac{\overline{(\rho v)' u' \frac{\partial \bar{H}}{\partial y}}}{\overline{(\rho v)' H' \frac{\partial \bar{u}}{\partial y}}} = \frac{c_p \epsilon}{\kappa} \quad (63)$$

The mean total enthalpy equation using this approach is as previously given (eq. (4) or 4(a)).

The second approach involves modeling the term

$$\overline{-(\rho v)' h'} = k^* \frac{\partial h}{\partial y}$$

which results in the static turbulent Prandtl number

$$N_{Pr,t} = \frac{\overline{(\rho v)' u' \frac{\partial h}{\partial y}}}{\overline{(\rho v)' h'} \frac{\partial u}{\partial y}} = \frac{c_p \epsilon}{k^*} \quad (64)$$

For this case the last term in the bracket of equation (4a) is modified to read

$$- \frac{1 - N_{Pr}}{1 - \zeta_w} \left(1 + \frac{\epsilon}{\mu} \frac{N_{Pr}}{N_{Pr,t}} \frac{1 - N_{Pr,t}}{1 - N_{Pr}} \right) \left(\frac{u_e^2}{2H_e} \frac{\partial F^2}{\partial y} + \frac{w_e^2}{2H_e} \frac{\partial g^2}{\partial y} \right)$$

The definition of M' (eq. (10e)) then becomes

$$M' = \frac{\varphi_r}{N_{Pr}} \frac{1 - N_{Pr}}{1 - \zeta_w} \left(1 + \frac{\epsilon}{\mu} \frac{N_{Pr}}{N_{Pr,t}} \frac{1 - N_{Pr,t}}{1 - N_{Pr}} \right) \quad (65)$$

For swept-leading-edge flows, the $N_{Pr,t}$ definition for turbulent Prandtl number must be used in order for the present system of equations to be conceptually correct. (See ref. 8.)

PROGRAM DESCRIPTION

The program is written in FORTRAN IV. A FORTRAN variable list is given first. Then, the various portions of the program, the main program and subroutines, each accompanied by flow charts, are given.

The main program controls the finite-difference solution of the boundary-layer equations, reads the input, calls the various subroutines, and writes the output. CALCM is called to compute the viscosity functions. During the iteration procedure, ABCDGS and COMPUTE are called successively for each variable to calculate the coefficients of the solution matrix and to compute new values for the variable, respectively. Library subroutines FTLUP and DISCOT are used for interpolation. A description of these subroutines is included in the appendix.

FORTRAN Variable List

AB value used from ABTAB

ABTAB table of A_b values in Van Driest damping function (see eq. (59))

ADL constant A_d in Van Driest damping function (A_d/L) (see eq. (58))

| | |
|------------------------------|--|
| AP | constant A in expression for \bar{f}_{\max} (see eq. (56a)) |
| BCG | temporary storage used in computing G and g in ABCDGS |
| BP | constant B in expression for \bar{f}_{\max} (see eq. (56a)) |
| CAPA CAPB CAPC CAPD | coefficients of solution matrix (see eqs. (25), (38), and (39)) |
| CAPG | |
| CAPMA2 | |
| CAPP1 | |
| CAPP2 | |
| CAPRS | reference Reynolds number, $\rho_s \sqrt{2H_e} L / \mu_s$ |
| CAPU1 | $\left[\frac{1}{u_e \sqrt{2H_e}} \frac{d(u_e / \sqrt{2H_e})}{d(x/L)} \right]_{m=1}$ (see eq. (10g)) |
| CAPU2 | $\left[\frac{1}{u_e \sqrt{2H_e}} \frac{d(u_e / \sqrt{2H_e})}{d(x/L)} \right]_{m=2}$ (see eq. 10g)) |
| CAPZ1 | $\left[\frac{1}{1 - \zeta_w} \frac{d\zeta_w}{d(x/L)} \right]_{m=1}$ (see eq. (10i)) |
| CAPZ2 | $\left[\frac{1}{1 - \zeta_w} \frac{d\zeta_w}{d(x/L)} \right]_{m=2}$ (see eq. (10i)) |
| CFF | wall-skin-friction coefficient for chordwise profile |

| | |
|---------|---|
| CFG | wall-skin-friction coefficient for spanwise profile |
| CKVAL | check value used in COMPUTE for left side of equations (35), (44), and (45) |
| CP | constant C in expression for \bar{f}_{\max} (see eq. (56a)) |
| DELDEL2 | temporary storage in equations for CAPA, CAPC, and CAPD in ABCDGS |
| DELETA | $\Delta\eta$ |
| DELXI | $\Delta\xi$ |
| DELXIO | initial $\Delta\xi$ step size |
| DSDLF | $(\delta^*/L)_F$ |
| DSDLG | $(\delta^*/L)_g$ |
| DUDXTAB | $\frac{d(u_e/\sqrt{2H_e})}{d(x/L)} \text{ table}$ |
| DUEDX1 | $\left[\frac{d(u_e/\sqrt{2H_e})}{d(x/L)} \right]_{m=1} \text{ (see eqs. (17b))}$ |
| DUEDX2 | $\left[\frac{d(u_e/\sqrt{2H_e})}{d(x/L)} \right]_{m=2} \text{ (see eqs. (17b))}$ |
| DUM2 | dummy in COMPUTE for F, g, and θ |
| DZDXTAB | $\frac{d\xi_w}{d(x/L)} \text{ table}$ |
| DZWDX1 | $\left[\frac{d\xi_w}{d(x/L)} \right]_{m=1} \text{ (see eqs. (17b))}$ |
| DZWDX2 | $\left[\frac{d\xi_w}{d(x/L)} \right]_{m=2} \text{ (see eqs. (17b))}$ |

| | |
|------------------|--|
| EMUSDMU | μ_s/μ |
| EPSDMU | ϵ/μ |
| EPSLONE | accuracy criteria for $\partial(F,G,\theta)/\partial\eta$ at outer edge of profile (σ_e) (see eqs. (35), (44), and (45)) |
| EPSLONW | convergence criteria for iterations on F , g , and θ profiles, allowable percent change in wall slope between iterations (σ_w) (see eqs. (46) to (48)) |
| ETA | η |
| ETATAB | input η table |
| FA FA2 } } | average values of F (see fig. 2) |
| FBAR | l/δ (see eq. (51)) |
| FBARMAX | maximum value of f |
| FBARTAB | \bar{f} values corresponding to YDDFB values |
| FCFTAB | $\frac{\rho_w v_w}{\rho_e u_e} \frac{2}{C_f}$ values corresponding to ABTAB values |
| FCT1 | $2K/(1 + K)$, used in ABCDGS to account for variable η step size |
| FCT2 | $2/(1 + K)$, used in ABCDGS to account for variable η step size |
| FDCFD2 | $\frac{\rho_w v_w}{\rho_e u_e} \frac{2}{C_f}$ |
| FNCDEL1 | intermediate variable used in calculation of H_i^* |
| FNCDEL2 | intermediate variable used in calculation of H_i^* |
| FNCFRR1 | intermediate variable used in calculation of $(\delta^*/L)_F$ |

| | |
|---------|--|
| FNCFRR2 | intermediate variable used in calculation of $(\delta^*/L)_F$ |
| FNCGRR1 | intermediate variable used in calculation of $(\delta^*/L)_g$ |
| FNCGRR2 | intermediate variable used in calculation of $(\delta^*/L)_g$ |
| FNCRER1 | intermediate variable used in calculation of y/L |
| FNCRER2 | intermediate variable used in calculation of y/L |
| FNCTHE1 | intermediate variable used in calculation of H_i^* |
| FNCTHE2 | intermediate variable used in calculation of H_i^* |
| FPREV | $F_{1,2}$ |
| FR0 | temperature recovery factor, used to compute Stanton number |
| FTAB | input u/u_e profile |
| FTEST | value of F at outer edge of boundary layer |
| FUNCF1 | intermediate variable used in calculation of $(\theta^*/L)_F$ |
| FUNCF2 | intermediate variable used in calculation of $(\theta^*/L)_F$ |
| FUNCG1 | intermediate variable used in calculation of $(\theta^*/L)_g$ |
| FUNCG2 | intermediate variable used in calculation of $(\theta^*/L)_g$ |
| FUNCX1 | intermediate variable used in calculation of table of ξ values, XITAB |
| FUNCX2 | intermediate variable used in calculation of table of ξ values, XITAB |
| FUNCY1 | intermediate variable used in calculation of table of η values, ETATAB |

| | |
|---------|--|
| FUNCY2 | intermediate variable used in computation of table of η values, ETATAB |
| F1 | $F_{1,n}$ |
| F2 | $F_{2,n}$ |
| GEE2 | input $g_{1,2}$ |
| GPREV | $g_{1,2}$ |
| GTAB | input w/w _e profile |
| GTEST | value of g at outer edge of boundary layer |
| G1 | $g_{1,n}$ |
| G2 | $g_{2,n}$ |
| HDCAPHE | h/H_e |
| HEDCPHE | h_e/H_e |
| HIS | $H_i^* = (\delta^*/\theta^*)_i$ (see eq. (57)) |
| HISF | $(H_i^*)_F$ |
| HISG | $(H_i^*)_g$ |
| HRDCPHE | h_e/H_e |
| HSHE | $1 - \frac{w_e^2}{2H_e}$ |
| I | a subscript |
| ICHD | indicates which D to compute in ABCDGS |
| ICOUNT | a count of number of times output is printed |

| | |
|---------|--|
| IFBLU | = 0 for computing \bar{f} = 1 for table lookup of \bar{f} |
| INIT | = 0 for $\Delta\xi_{\text{initial}} = \text{DELXIO}$ = 1 for small $\Delta\xi_{\text{initial}}$ |
| ITERATE | indicates number -1 of iterations to get an acceptable set of values for all variables at a particular step |
| ITHETA | code for input temperature profile ITHETA = 1 if RHOTAB used ITHETA = 2 if ZETATAB used |
| IUSEEMU | = 0 for laminar ($\epsilon/\mu = 0$) solution = 1 for turbulent solution |
| IVAL | a variable used to denote outer boundary (number of equations (or $\Delta\eta$ steps) to be used at a particular step for a particular variable) |
| IVEG | = 1 for printout of V profile = 2 for printout of ϵ/μ profile = 3 for printout of g profile |
| IWLDMMP | = 0 for using wall properties in wall damping function = 1 for using local properties in wall damping function |
| J | body shape index j = 0 for two-dimsional flows j = 1 for axisymmetric flows |
| JJ | a subscript used in COMPUTE |
| KF | a subscript used in COMPUTE |
| KMAX | N-10 (Number of values computed across boundary layer - 10) |
| M | denotes order of interpolation for library subroutine FTLUP |

| | |
|---------|--|
| MAX | indicates number of values to be printed |
| MPWEMU | = 0 for PRTTAB = $N_{Pr,T}$ values = 1 for PRTTAB = $N_{Pr,t}$ values |
| M2 | -2, indexing parameter in COMPUTE |
| NBACK | indexing parameter used in COMPUTE to denote edge |
| NF | subscript used in COMPUTE |
| NFBY | number of values in FBARTAB |
| NFCFAB | number of values in ABTAB |
| NMAXF | number of η steps to outer edge of F profile |
| NMAXG | number of η steps to outer edge of g profile |
| NMAXTH | number of η steps to outer edge of θ profile |
| NPRINT | counter used to determine when to print |
| NSTEPS | number of ξ steps between profile printouts |
| NUMDELE | NUMETA - 1 |
| NUMETA | maximum number of steps in η -direction |
| NUMX | number of values in XL table |
| NUMY | number of values in YL table |
| NYP | number of values in PRTTAB |
| OL | reference length |
| PHIR | $(\rho\mu)/(\rho\mu)_e$ (see eq. (19)) |

| | |
|---------|---|
| PR | molecular Prandtl number |
| PRT | turbulent Prandtl number |
| PRTTAB | turbulent Prandtl number table |
| QBAR | output heat-transfer parameter, $\frac{\dot{q}_w L}{\mu_e H_e}$ |
| R | r, body radius divided by L |
| RA | average value of r |
| RCURJ | a combination of terms used in computing $\bar{\theta}/L$ |
| RDXL | $R_{e,x}/L$ |
| REDSF | $\frac{\rho_e u_e \delta^*_F}{\mu_e}$ |
| REDSG | $\frac{\rho_e u_e \delta^*_g}{\mu_e}$ |
| RERS | ρ_e/ρ_s |
| RERSA | $(\rho_e/\rho_s)_a$ |
| RERSTAB | ρ_e/ρ_s table |
| RERS1 | $(\rho_e/\rho_s)_1$ |
| RERS2 | $(\rho_e/\rho_s)_2$ |
| RETSF | $\frac{\rho_e u_e \theta^*_F}{\mu_e}$ |
| RETSG | $\frac{\rho_e u_e \theta^*_g}{\mu_e}$ |
| REX | $\frac{\rho_e u_e x}{\mu_e}$ |
| RHOEROA | $(\rho_e/\rho)_a$ |

| | |
|---------|--|
| RHOERO1 | $(\rho_e/\rho)_{1,n}$ |
| RHOERO2 | $(\rho_e/\rho)_{2,n}$ |
| RHORHOE | ρ/ρ_e |
| RHOTAB | input ρ_e/ρ table |
| RMRRMSA | average value of $(\rho\mu)_e/(\rho\mu)_s$ |
| RMUTAB | table of $(\rho\mu)_e/(\rho\mu)_s$ |
| RRUUER | $R_s \frac{(\rho\mu)_e}{(\rho\mu)_s} \frac{u_e}{\sqrt{2H_e}} r^{2j}$ |
| RTAB | r/l table |
| RURDRUS | $(\rho\mu)_e/(\rho\mu)_s$ |
| RUSDRUR | $(\rho\mu)_s/(\rho\mu)_e$ |
| R1 | r_1 |
| R2 | r_2 |
| SHE | Sutherland's constant divided by H_e |
| SMDEL | summation used in computing H_i^* |
| SMLG | coefficient in formula for dependent variables (see eq. (31), for example) |
| SMRER | summation used in computing y/δ |
| SMTHE | summation used in computing H_i^* |
| ST | Stanton number |
| SUMF | summation used in computing $(\theta^*/L)_F$ |
| SUMFETA | summation used in computing η |

| | |
|---------|---|
| SUMFRR | summation used in computing $(\delta^*/L)_F$ |
| SUMFXI | summation used in computing ξ |
| SUMG | summation used in computing $(\theta^*/L)_g$ |
| SUMGRR | summation used in computing $(\delta^*/L)_g$ |
| SUMRER | summation used in computing y/L and y/δ in CALCM |
| TAFDTAG | ratio of chordwise to spanwise shear at wall |
| TEST | value of F , g , or θ at edge of boundary layer |
| THETAA | average value of θ |
| THETAPR | $\theta_{1,2}$ |
| THETA1 | $\theta_{1,n}$ |
| THETA2 | $\theta_{2,n}$ |
| THTEST | value of θ at edge of boundary layer |
| TIMES | temporary storage used in computing V_a |
| TMDLDL2 | temporary storage used in computing CAPB and CAPD in ABCDGS |
| TSDLF | $(\theta^*/L)_F$ |
| TSDLG | $(\theta^*/L)_g$ |
| TSTVAL | test value used in COMPUTE for right side of equations (35), (44), and (45) |
| UEDSTAB | $u_e / \sqrt{2H_e}$ |
| UES2HEA | $(u_e / \sqrt{2H_e})_a$ |
| UES2HE1 | $(u_e / \sqrt{2H_e})_1$ |

| | |
|---------|---|
| UES2HE2 | $(u_e / \sqrt{2H_e})_2$ |
| VA | average values of V |
| VADLDL2 | temporary storage used in computing CAPA and CAPC in ABCDGS |
| VWA | average value of VWTAB |
| VWTAB | v_w/u_e for axisymmetric or two-dimensional flow v_w/w_e for swept-leading-edge flow |
| WEDS2HE | $w_e / \sqrt{2H_e}$ |
| XI | ξ |
| XIA | ξ_a |
| XIBARA | $(2\xi)_a^{2\bar{n}}$ |
| XIBAR1 | $(2\xi)_1^{2\bar{n}}$ |
| XIBAR2 | $(2\xi)_2^{2\bar{n}}$ |
| XIBCPPA | $(\bar{\xi}P)_a$ |
| XIBCPUA | $(\bar{\xi}U)_a$ |
| XIBCPZA | $(\bar{\xi}Z)_a$ |
| XIDELFA | temporary storage used in computing CAPB and CAPD in ABCDGS |
| XINB | $(2\xi)^{\bar{n}}$ |
| XISTOP | value of ξ where solution is to be terminated |
| XITAB | table of ξ values where input is specified |
| XITEST | value of ξ where $\Delta\xi$ is increased by a factor of 10 |

| | |
|---------|---|
| XI0 | value of ξ at input station |
| XI1 | ξ_1 |
| XI2 | ξ_2 |
| XK | $\Delta\eta_n/\Delta\eta_{n-1}$ |
| XL | table of x/L values |
| XLPR | x/L values where profile printout is required |
| XMBAR | \bar{M} |
| XMBARA | \bar{M}_a |
| XMBARA2 | $\bar{M}_a/2$ |
| XMBAR1 | $\bar{M}_{1,n}$ |
| XMBAR2 | $\bar{M}_{2,n}$ |
| XMCIRA2 | $\hat{M}_a/2$ |
| XMCIRC | \hat{M} |
| XMCIRCA | \hat{M}_a |
| XMCIRC1 | $\hat{M}_{1,n}$ |
| XMCIRC2 | $\hat{M}_{2,n}$ |
| XMME F | $(M/M_e)_F$ |
| XMMEG | $(M/M_e)_g$ |

(see eqs. (10b) and (10c))

| | | |
|----------|--|---|
| XMPRIM | M' | M' M'_a $M'_{1,n}$ $M'_{2,n}$ $M'_{a/2}$ M^* M_a^* $M_{1,n}^*$ $M_{2,n}^*$ $M_{a/2}^*$ |
| XMPRIMA | M'_a | |
| XMPRIM1 | $M'_{1,n}$ | |
| XMPRIM2 | $M'_{2,n}$ | |
| XMPRMA2 | $M'_{a/2}$ | |
| XMSTAR | M^* | |
| XMSTARA | M_a^* | |
| XMSTAR1 | $M_{1,n}^*$ | |
| XMSTAR2 | $M_{2,n}^*$ | |
| XMSTRA2 | $M_{a/2}^*$ | |
| XNBAR | \bar{n} | |
| XXL | x/L | |
| $x_0 \}$ | initial value of x/L | |
| $x_1 \}$ | | |
| YDD | y/δ | |
| YDDFB | y/δ | values corresponding to FBARTAB |
| YDDPRT | y/δ | values corresponding to PRTTAB |
| YDL | y/L | |
| YL | table of y/L values for initial profiles | |
| ZETA | ζ | |

ZETAE $\zeta_e = 1.0$

ZETATAB input ζ profile

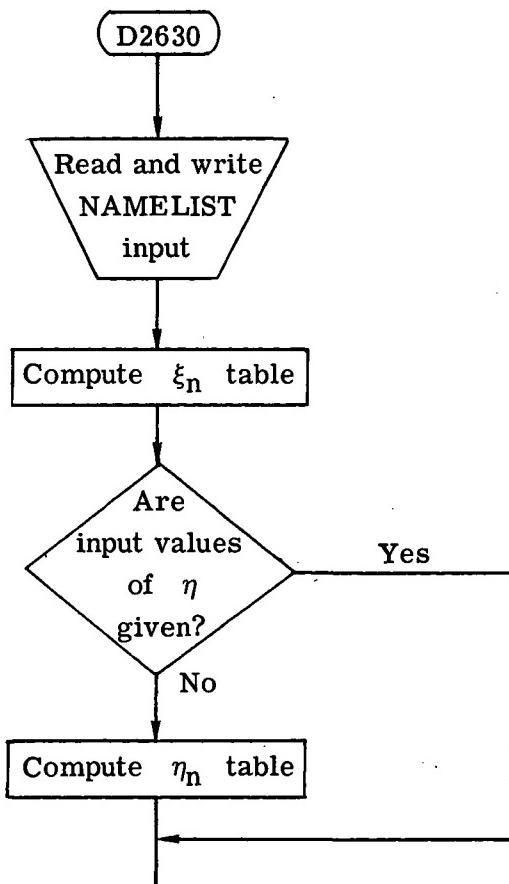
ZETAW1 ζ_{w_1}

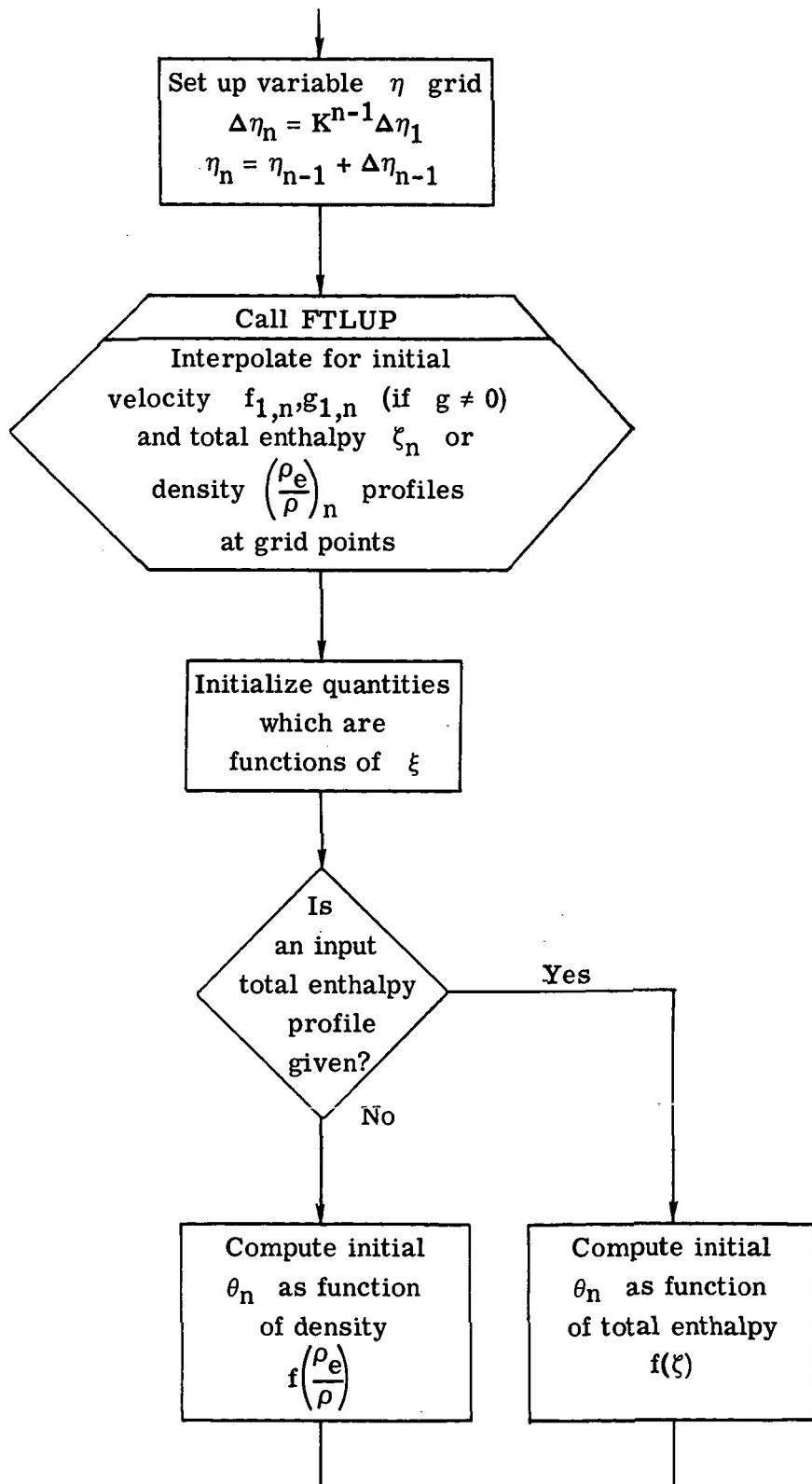
ZETAW2 ζ_{w_2}

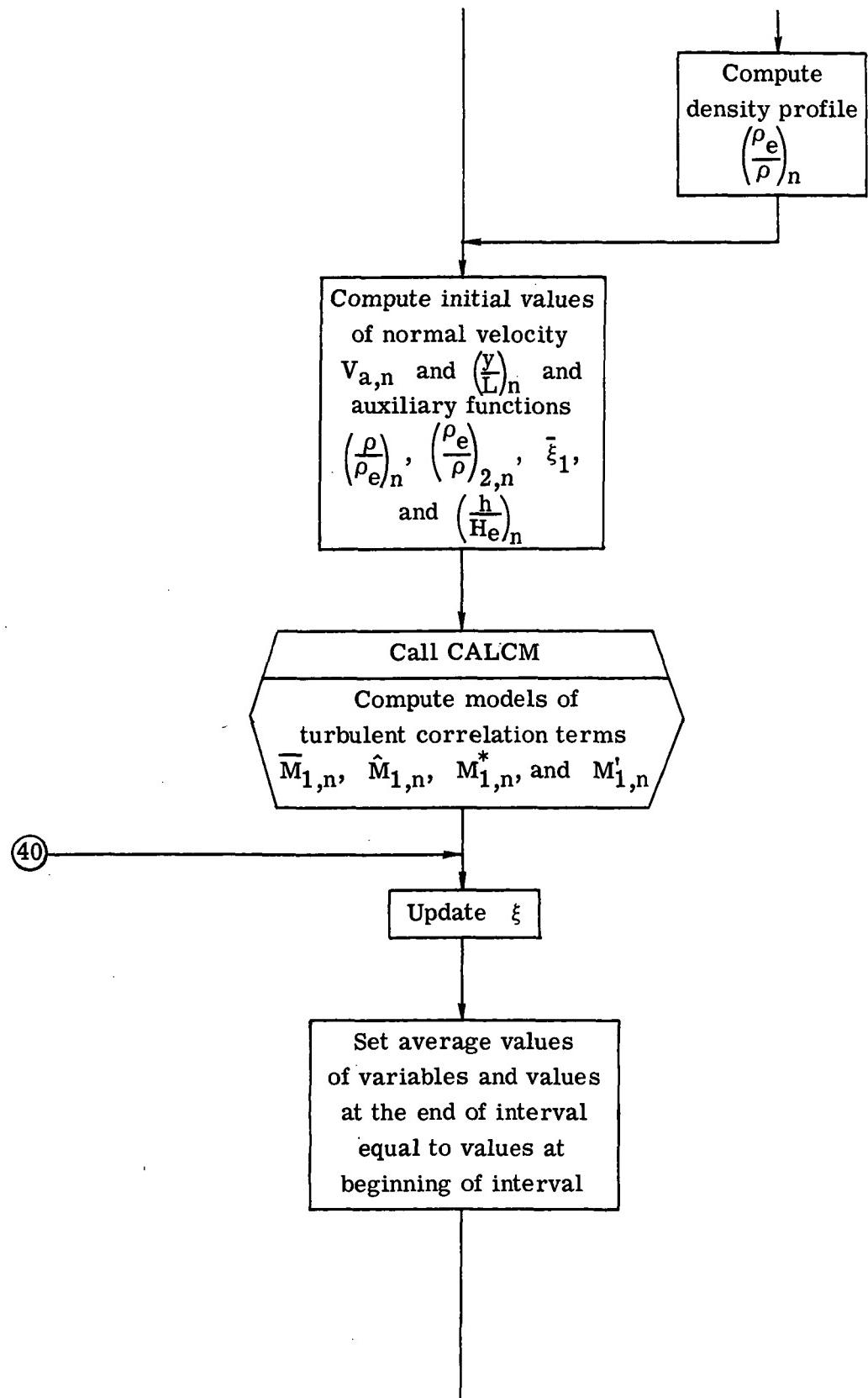
ZETWTAB ζ_w table

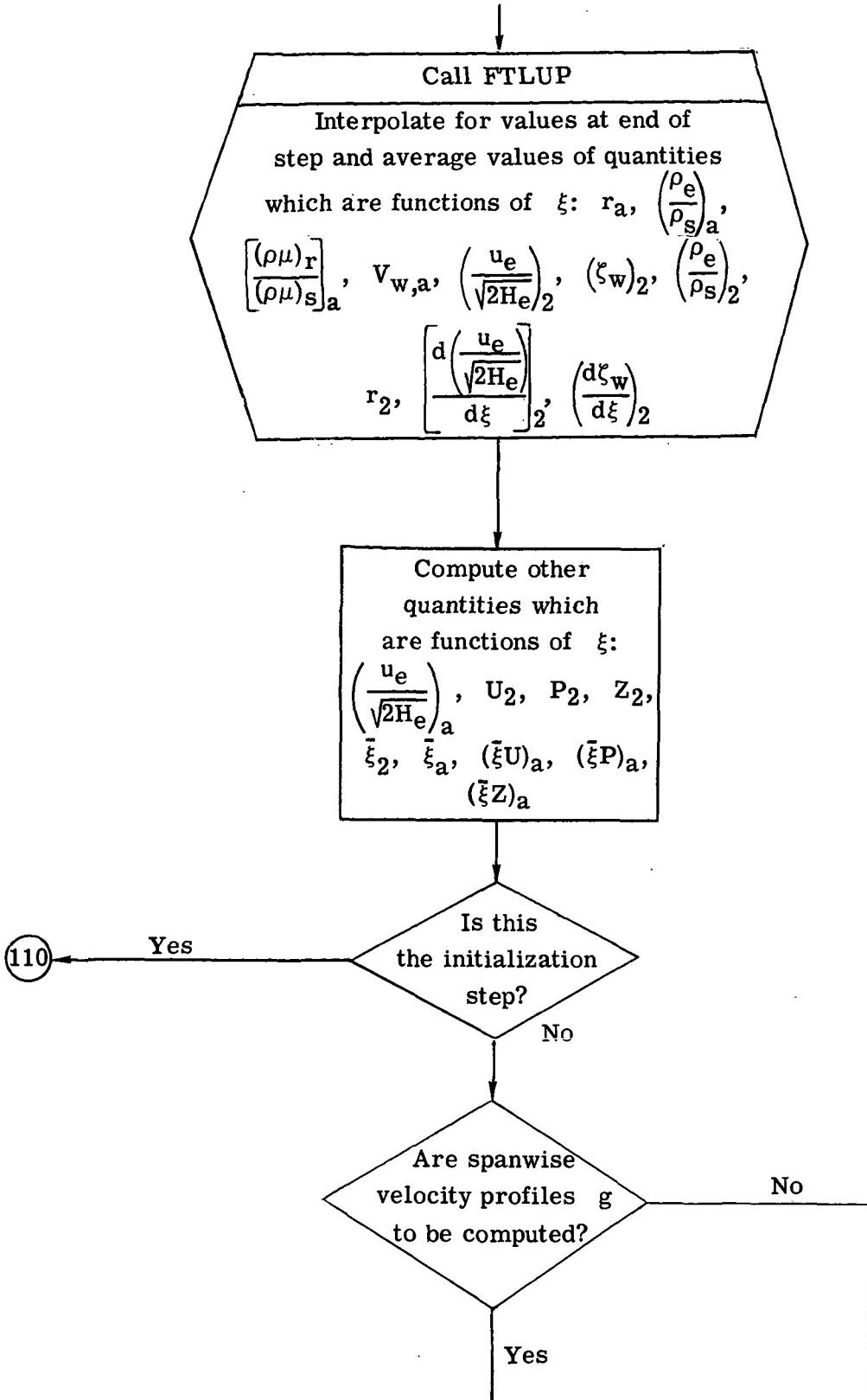
Description, Flow Charts, and Listings of the
Main Program and Subprograms

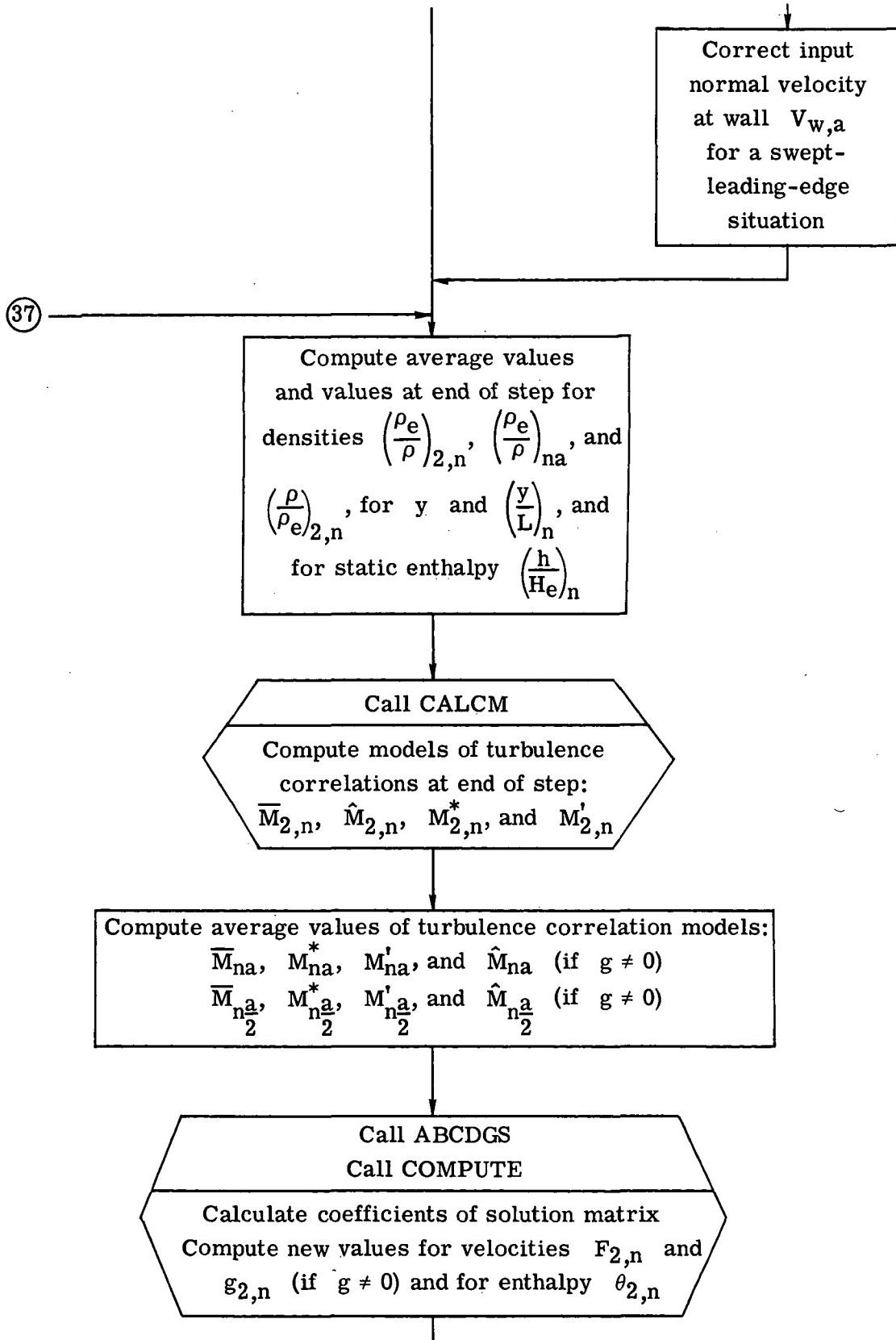
Main program D2630.- The main program controls the finite-difference solution of the boundary-layer equations. It reads the input, computes other initial conditions, sets up the grid used in the solution, calls all the subroutines, and writes the output. The flow-diagram of main program D2630 is as follows:

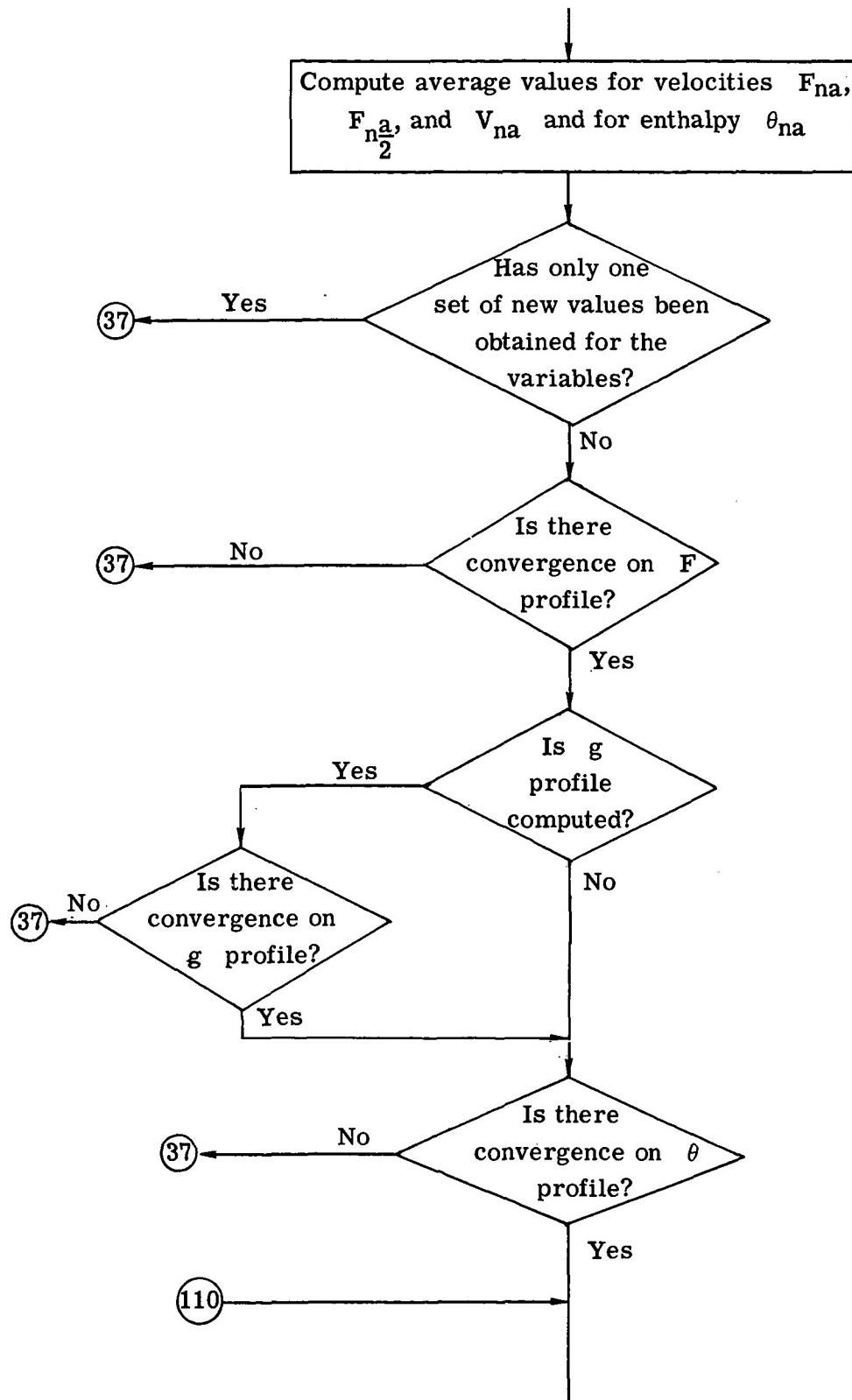


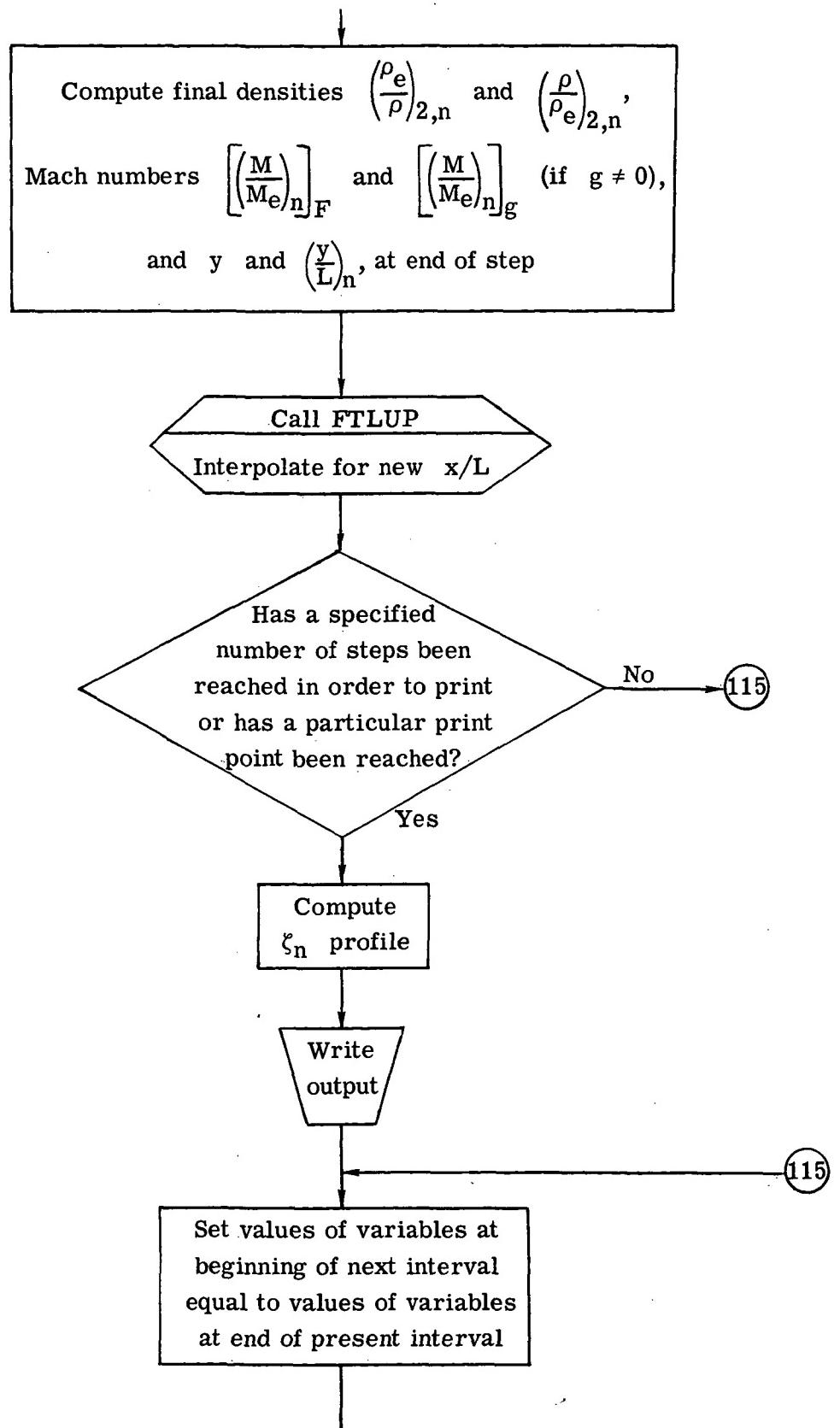


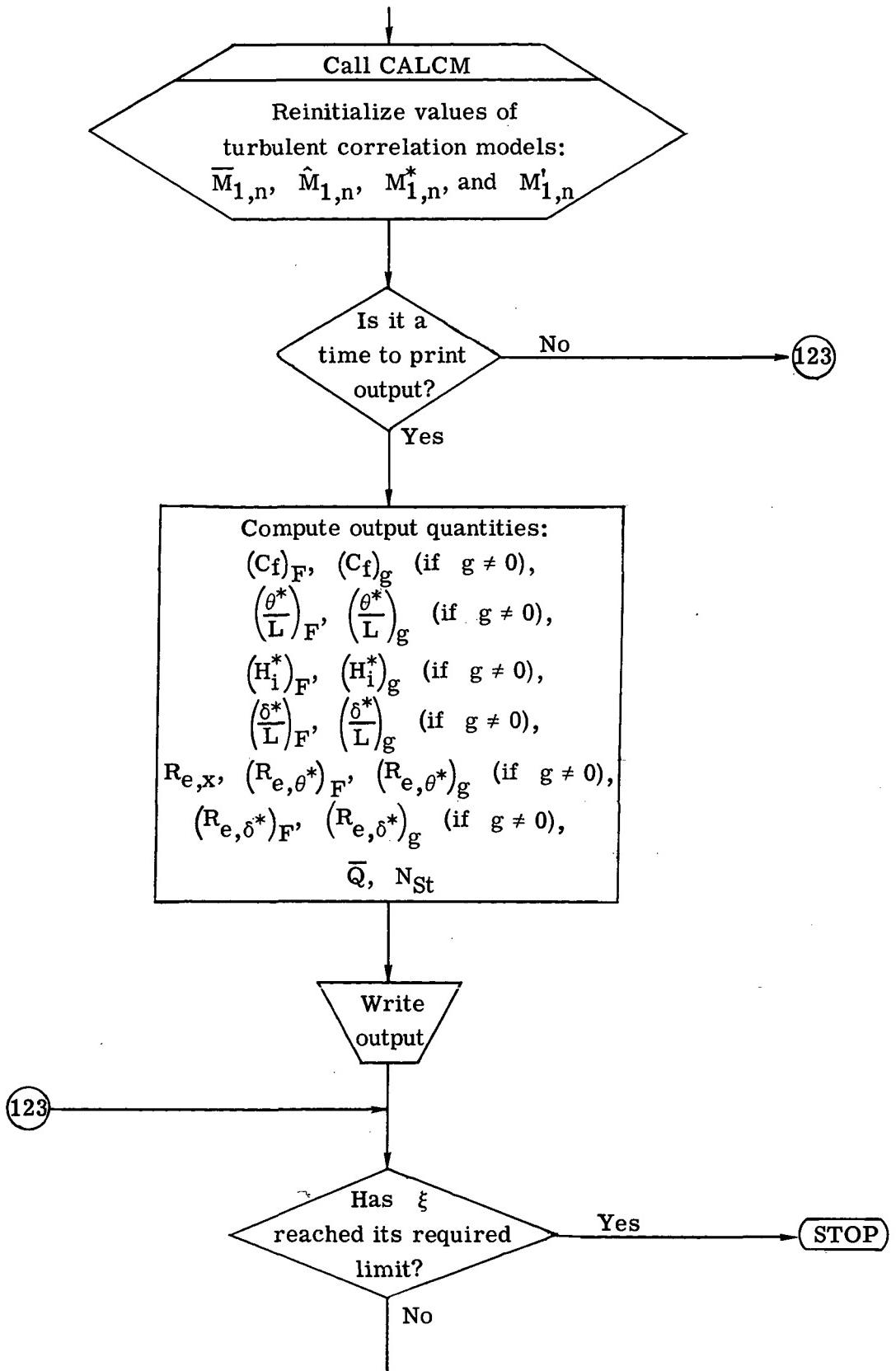


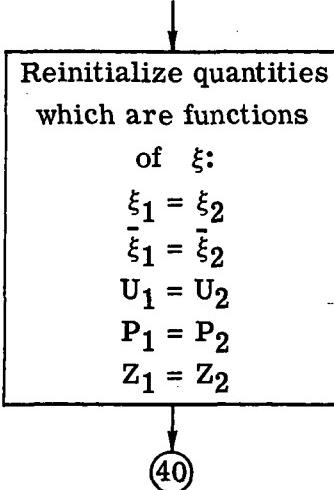












The program listing for Main program D2630 is as follows:

```

PROGRAM D2630(INPUT,OUTPUT,TAPE5=INPUT,TAPE6=OUTPUT)
DIMENSION DELETA(350),FTAB(100),F1(350),FA(350),FA2(350),
1VA(350),GTAB(100),G1(350),G2(350),THETA1(350),THETA2(350),
2THETAA(350),RHOTAB(100),RHOERO1(350),RHOEROA(350),ZETATAB(100),
3ZETA(350),XMBAR1(350),XMCIRC1(350),XMSTAR1(350),XMPRIM1(350),
4PHIR(350),XMBARA2(350),XMCIRA2(350),XMSTRA2(350),XMPRMA2(350),
5CAPG(350),SMLG(350),ETATAB(100),DUDXTAB(75),DZDXTAB(75),
6XITAB(75),XMBAR2(350),XMCIRC2(350),XMSTAR2(350),XMPRIM2(350),
7XMBARA(350),XMCIRCA(350),XMSTARA(350),XMPRIMA(350),SUMRER(350),
8F2(350),XLPR(30),VWTAB(75),RMUTAB(75),ABTAB(20),FCFTAB(20),
9XMMEG(350),XMMEF(350),ETA(350),UEDSTAB(75),ZETWTAB(75),XL(75),
1RTAB(75),RERSTAB(75),YL(100)
COMMON/EPSDMU/EPSDMU(350)
COMMON/TABLE1/RHOERO2(350)
COMMON/TABLE2/YDL(350),RHORHOE(350)
COMMON/THREE/NUMETA,NMAXF
COMMON/FEB12/VWA
COMMON/FEB11/PRTTAB(20),YDDPRT(20),NYP
COMMON/HDCAPHE/HDCAPHE(350),GEE2
COMMON/MUUSE/IUSEEMU,MPWEMU
COMMON/FBAR/FBARTAB(20),IFBLU,YDDFB(20),NFBY
COMMON/IWLDMPI/WLDMPI
NAMELIST/NAM1/NUMETA,NMAXF,NMAXG,DELETA,XK,XIO,DELXIO,XITEST,
1XISTOP,FTAB,ETATAB,VWTAB,EPSLONE,EPSLONW,          GTAB,UEDSTAB,
2 WEDS2HE,PR,ZETWTAB,XNBAR,RERSTAB,CAPRS,RTAB,J,RHOTAB,
3SHE,ZETATAB,I THETA,HSHE,XL,NUMX,YL,NUMY,
4XO,OL,DUDXTAB,DZDXTAB,FRO,NSTEPS,
5AP,BP,CP,ABTAB,FCFTAB,NFCFAB,PRTTAB,YDDPRT,NYP,
6           XLPR,IVEG,INIT,IUSEEMU,MPWEMU,
7FBARTAB,IFBLU,YDDFB,NFBY,IWLDMPI
ZETATAB(2)=0.0
RHOTAB(2)=0.0
ETATAB(2)=0.0
XLPR(2)=0.0
XIO=0.0
NPRINT=1
ICOUNT=1
READ(5,NAM1)
  
```

```

      WRITE(6,NAM1)

C      OPTION FOR INITIAL DELTA XI

C      IF(INIT.EQ.0)DELXI=DEFLXIO
C      IF(INIT.EQ.1)DELXI=X10*1.E-5
C      GEE2=GTAB(2)

C      T R A N S F O R M   X   T O   X I

C      HRDCPHE=1.-UEDSTAB(1)**2-WEDS2HE**2
C      RMUTAB(1)=SQRT(HRDCPHE/HSHE)*(HSHE+SHE)/(HRDCPHE+SHE)
C      1*RERSTAB(1)*HRDCPHE
C      2/HSHE
C      FUNCX1=RMUTAB(1)*UEDSTAB(1)*RTAB(1)**(2*j)
C      X1=X0
C      DO 22 I=1,NUMX
C      IF(I.EQ.1)GO TO 15
C      HRDCPHE=1.-UEDSTAB(I)**2-WEDS2HE**2
C      RMUTAB(I)=SQRT(HRDCPHE/HSHE)*(HSHE+SHE)/(HRDCPHE+SHE)
C      1*RERSTAB(I)*HRDCPHE
C      2/HSHE
C      FUNCX2=RMUTAB(I)*UEDSTAB(I)*RTAB(I)**(2*j)
C      SUMFXI=SUMFXI+(FUNCX2+FUNCX1)/2.* (XL(I)-XL(I-1))
C      FUNCX1=FUNCX2
C      GO TO 95
15  SUMFXI=0.
95  RRUUER=CAPRS*RMUTAB(I)*UEDSTAB(I)*RTAB(I)**(2*j)
     IF(X10.EQ.0.)GO TO 25
     GO TO 31
25  X10=RRUUER*X1
31  X1TAB(I)=X10+CAPRS*S1(MFXI
     DUDXTAB(I)=DUDXTAB(I)/RRUUER
     DZDXTAB(I)=DZDXTAB(I)/RRUUER
22  CONTINUE
     X11=X10

C      I F   N O   E T A   T A B L E ,   T R A N S F O R M   Y   T O   E T A

C      IF(ETATAB(2).NE.0.)GO TO 26
C      IF(ZETATAB(2).EQ.0.)GO TO 32
C      DO 202 I=1,NUMY
C      THETA1(I)=(ZETATAB(I)-ZETWTAB(I))/(1.-ZETWTAB(I))
202  CONTINUE
     DO 33 I=1,NUMY
     RHOTAB(I)=((1.-ZETWTAB(I))*THETA1(I)+ZETWTAB(I)-UEDSTAB(I)**2
     1*FTAB(I)**2-WEDS2HE **2*GTAB(I)**2)/(1.-UEDSTAB(I)**2-WEDS2HE
     **2)
33  CONTINUE
32  CONTINUE
     DO 34 I=1,NUMY
     RHORHOE(I)=1./RHOTAB(I)
34  CONTINUE
     FUNCY1=RHORHOE(I)*RERSTAB(1)
     SUMFFTA=0.0
     ETATAB(1)=0.0
     DO 27 I=2,NUMY
     FUNCY2=RHORHOE(I)*RERSTAB(1)
     SUMFETA=SUMFETA+(FUNCY2+FUNCY1)/2.* (YL(I)-YL(I-1))

```

```

FUNCY1=FUNCY2
ETATAB(I)=CAPRS*UEDSTAB(1)*RTAB(1)**J/(2*X10)**XNBAR *SUMFETA
27 CONTINUE
26 CONTINUE
C
C      C O N S T A N T S
C
ZETAE=1.
FTEST=.99999
GTEST=.99999
THTEST=.99999
ETA(1)=0.
SUMRER(1)=0.

C
C      C O M P U T E   D F L T A   E T A S
C
F1(1)=FTAB(1)
G1(1)=GTAB(1)
IF(ZETATAB(2).NE.0.)ZETA(1)=ZETATAB(1)
IF(RHOTAB(2).NE.0.)RHOERO1(1)=RHOTAB(1)
M=2
DO 10 I=2,NUMETA
DELETA(I)=XK***(I-1)*DELETA(I)

C
C      C O M P U T E   E T A   S
C      I N T E R P O L A T E   T O   G E T   F ( E T A )   S
C
ETA(I)=ETA(I-1)+DELETA(I-1)
IF(F1(I-1).GT..9)M=1
CALL FTLUP(ETA(I),F1(I),M,NUMY,ETATAB,FTAB)
M=2
IF(GTAB(2).EQ.0.)GO TO 17
IF(G1(I-1).GT..9)M=1
CALL FTLUP(ETA(I),G1(I),M,NUMY,ETATAB,GTAB)
M=2
GO TO 16
17 G1(I)=0.0
16 IF(ZETATAB(2).EQ.0.)GO TO 18
IF(ZETA(I-1).GT..9)M=1
CALL FTLUP(ETA(I),ZETA(I),M,NUMY,ETATAB,ZETATAB)
M=2
GO TO 19
18 IF(ABS(RHOERO1(I-1)-1.).GE..1)M=1
CALL FTLUP(ETA(I),RHOERO1(I),M,NUMY,ETATAB,RHOTAB)
19 CONTINUE
10 CONTINUE
C
C      I N I T I A L   F ( X I 1 )   S
C
UES2HE1=UEDSTAB(1)
ZETAW1=ZETWTAB(1)
RERS1=RERSTAB(1)
R1=RTAB(1)
DUDDX1=DUDXTAB(1)
DZwdx1=DZDXTAB(1)
CAPU1=DUDDX1/UES2HE1
CAPP1=-CAPU1
CAPZ1=DZwdx1/(1.-ZETAW1)

```

```

C
C      C O M P U T E I N I T I A L   T H E T A S
C
C      IF (ITHETA.EQ.2)GO TO 11
C      THETA1(1)=0.0
C      DO 200 I=2,NUMETA
C      THETA1(I)=(RHOERO1(I)*(1.-UES2HE1**2-WEDS2HE**2)-ZETAW1+UES2HE1
C      1**2*F1(I)**2+WEDS2HE**2*G1(I)**2)/(1.-ZETAW1)
C 200 CONTINUE
C      GO TO 12
C 11 DO 201 I=1,NUMETA
C      THETA1(I)=(ZETA(I)-ZETAW1)/(1.-ZETAW1)
C 201 CONTINUE
C 12 CONTINUE
C
C      I F Z E T A S A R E G I V E N , I N I T I A L   R H O E R O 1 S
C      M U S T B E C A L C U L A T E D
C
C      IF(ZETATAB(2).EQ.0.)GO TO 28
C      DO 29 I=1,NUMETA
C      RHOERO1(I)=((1.-ZETAW1)*THETA1(I)+ZETAW1-UES2HE1**2*F1(I)**2
C      1-WEDS2HE**2*G1(I)**2)/(1.-UES2HE1**2-WEDS2HE**2)
C 29 CONTINUE
C 28 CONTINUE
C      DO 21 I=1,NUMETA
C      RHORHOE(I)=1./RHOERO1(I)
C 21 RHOERO2(I)=RHOERO1(I)
C
C      C O M P U T E I N I T I A L   V S
C
C      XIBAR1=(2.*X10)**(2.*XNBAR )
C      VA(1)=(2*X11)**XNBAR /R1**J*RHORHOE(1)*RERS1*VWTAB(1)/RMUTAB(1)
C      IF(GEE2.NE.0.)VA(1)=VA(1)*WFDS2HE/UES2HE1
C      DO 20 I=2,NUMETA
C      VA(I)=VA(I-1)-DELETA(I-1)*XIBAR1*(F1(I)+F1(I-1))*XNBAR /(2.*X10)
C 20 CONTINUE
C      YDL(1)=0.0
C      FNCRER1=RHOERO2(1)
C      DO 42 I=2,NUMETA
C      FNCRER2=RHOERO2(I)
C      SUMRER(I)=SUMRER(I-1)+(FNCRER2+FNCRER1)/2.*DELETA(I-1)
C      FNCRER1=FNCRER2
C      YDL(I)=(2.*X11)**XNBAR *i./RERS1*SUMRER(I)/(CAPRS*R1**J*UES2HE1)
C 42 CONTINUE
C
C      C A L C U L A T E I N I T I A L   M S
C
C      VWA=VWTAB(1)
C      IF(GEE2.NE.0.)VWA=VWA*WEDS2HE/UES2HE1
C      DO 44 I=1,NUMETA
C      HDCAPHE(I)=(1.-ZETAW1)*THETA1(I)+ZETAW1-UES2HE1**2*F1(I)**2
C      1-WEDS2HE**2*G1(I)**2
C 44 CONTINUE
C      CALL CALCM(ZETAW1,UES2HE1,F1,WEDS2HE,ZETA,E,
C      1           SHE,PR,     XMBAR1,XMCIRC1,XMSTAR1,XMPRIM1,PHIR,
C      2HSHE,        RERS1,X11,XNBAR ,R1,J,
C      3DELETA,RURDRUS,    CAPRS,SUMRER,AP,BP,CP,       ABTAB,FCFTAB,NFCFAB,
C      4NMAXG,G1)
C 40 CONTINUE

```

```

C
C      B E G I N   N E W   I N T E R V A L
C
C      IF(XI1.GE.XI0+30.*DELXI.AND.INIT.EQ.1)GO TO 46
C      GO TO 48
46  DELXI=DELX10
    INIT=0
48  CONTINUE
    IF(XI1.LT.XTEST)GO TO 35
    DELXI=DELXI*10.
    XTEST=XTEST*10.
35  XI2=XI1+DELXI
    IF(ICOUNT.EQ.1)XI2=XI1
    XIA=(XI1+XI2)/2.
    ITERATE=1
    DO 30 I=1,NUMETA
    F2(I)=F1(I)
    FA(I)=F1(I)
    G2(I)=G1(I)
    THETA2(I)=THETA1(I)
    THETAA(I)=THETA1(I)
    XMBAR2(I)=XMBAR1(I)
30  CONTINUE
52  CONTINUE
C
C      I N T E R P O L A T E   F O R   F ( X I 2 ) S
C
C      CALL FTLUP(XIA,RA,1,NUMX,XITAB,RTAB)
C      CALL FTLUP(XIA,RERSA,1,NUMX,XITAB,RERSTAB)
C      CALL FTLUP(XIA,RMRRMSA,1,NUMX,XITAB,RMUTAB)
C      CALL FTLUP(XIA,VWA,1,NUMX,XITAB,VWTAB)
C      CALL FTLUP(XI2,UES2HF2,1,NUMX,XITAB,UEDSTAB)
C      CALL FTLUP(XI2,ZETAW2,1,NUMX,XITAB,ZETWTAB)
C      CALL FTLUP(XI2,RERS2,1,NUMX,XITAB,RERSTAB)
C      CALL FTLUP(XI2,R2,1,NUMX,XITAB,RTAB)
C      CALL FTLUP(XI2,DUEDX2,1,NUMX,XITAB,DUDXTAB)
C      CALL FTLUP(XI2,DZWDX2,1,NUMX,XITAB,DZDXTAB)
C      UES2HEA=(UES2HE1+UES2HE2)/2.
C      CAPU2=DUEDX2/UES2HE2
C      CAPP2=-CAPU2
C      CAPZ2=DZWDX2/(1.-ZETAW2)
C      XIBAR2=(2.*XI2)**(2.*XNBAR)
C      XIBARA=(XIBAR1+XIBAR2)/2.
C      XIBCPUA=(XIBAR1*CAPU1+XIBAR2*CAPU2)/2.
C      XIBCPPA=(XIBAR1*CAPP1+XIBAR2*CAPP2)/2.
C      XIBCPZA=(XIBAR1*CAPZ1+XIBAR2*CAPZ2)/2.
C      IF(ICOUNT.EQ.1)GO TO 110
C      IF(GEE2.NE.0.)VWA=VWA*WEDS2HE/UES2HEA
C
C      I T E R A T E   W I T H   S A M E   X I 2
C
37  DO 36 I=1,NUMETA
    RHOERO2(I)=((1.-ZETAW2)*THETA2(I)+ZETAW2-UES2HE2**2*F2(I)**2
    -WEDS2HE**2*G2(I)**2)/(1.-UES2HE2**2-WEDS2HE**2)
    RHOEROA(I)=(RHOERO1(I)+RHOERO2(I))/2.
    RHORHOE(I)=1./RHOERO2(I)
36  CONTINUE
    FNCRER1=RHOERO2(1)
    DO 38 I=2,NUMETA

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```

FNCRER2=RHOERO2(I)
SUMRER(I)=SUMRER(I-1)+(FNCRER2+FNCRER1)/2.*DELETA(I-1)
FNCRER1=FNCRER2
YDL(I)=(2.*X12)**XNRAR *1./RERS2*SUMRER(I)/(CAPRS*R2**J*UES2HE2)
38 CONTINUE

C A L C U L A T E M S

DO 39 I=1,NUMETA
HDCAPHE(I)=(1.-ZETAW2)*THETA2(I)+ZETAW2-UES2HE2**2*F2(I)**2
1-WEDS2HE**2*G2(I)**2
39 CONTINUE
CALL CALCM(ZETAW2,UES2HE2,F2,WEDS2HE,ZETAE,
1 SHE,PR,XMBAR2,XMCIRC2,XMSTAR2,XMPRIM2,PHIR,
2HSHE,RERS2,X12,XNBAR,R2,J,
3DELETA,RURDRUS,CAPRS,SUMRER,AP,BP,CP,ABTAB,FCFTAB,NFCFAB,
4NMAXG,G2)
DO 54 I=1,NUMETA
XMBARA(I)=(XMBAR1(I)+XMBAR2(I))/2.
IF(GEE2.NE.0.)XMCIRCA(I)=(XMCIRC1(I)+XMCIRC2(I))/2.
XMSTARA(I)=(XMSTAR1(I)+XMSTAR2(I))/2.
XMPRIMA(I)=(XMPRIM1(I)+XMPRIM2(I))/2.
54 CONTINUE
NUMDFLE=NUMETA-1
DO 50 I=1,NUMDELE
XMBARA2(I)=(XMBARA(I+1)+XMBARA(I))/2.
IF(GEE2.NE.0.)XMCIRRA2(I)=(XMCIRCA(I+1)+XMCIRCA(I))/2.
XMSTRA2(I)=(XMSTARA(I+1)+XMSTARA(I))/2.
XMPRMA2(I)=(XMPRIMA(I+1)+XMPRIMA(I))/2.
50 CONTINUE

C A L C U L A T E F S

ICHD=1
FPREV=F2(2)
55 CALL ABCDGS(NUMDELE,DELETA,XIBARA,DELXI,FA,VA,XMBARA2,ICHDF1,
1XIBCPUA,XIBCPPA,RHOEROA,G1,THETA1,XIBCPZA,THETAA,UES2HEA,XMPRMA2,
2F2,WEDS2HE,G2,CAPG,SMLG,XK)
CALL COMPUTE(NMAXF,FTEST,NUMDELE,DELETA,EPSSLONE,CAPG,SMLG,F2)

U P D A T E F A S A N D V A S

DO 70 I=2,NUMETA
FA(I)=(F1(I)+F2(I))/2.
70 CONTINUE
DO 73 I=1,NUMDFLE
FA2(I)=(FA(I+1)+FA(I))/2.
73 CONTINUE
TIMES=XNBAR/XIA
VA(1)=(2*XIA)**XNBAR /RA**J/RHOEROA(1)*RERSA*VWA/RMRRMSA
DO 74 I=2,NUMETA
VA(I)=VA(I-1)-DELETA(I-1)*XIBARA/(2.*DELXI)*(F2(I)+F2(I-1)-F1(I)
1-F1(I-1))-DELETA(I-1)*XIBARA*FA2(I-1)*TIMES
74 CONTINUE

C A L C U L A T E G S

IF(GTAB(2).EQ.0.)GO TO 75
ICHDF2

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GPREV=G2(2)
CALL ABCDGS(NUMDELE,DELETA,XIBARA,DELXI,FA,VA,XMC1RA2,ICHDR,F1,
1XIBCPUA,XIBCPPA,RHOEROA,G1,THETA1,XIBCPZA,THETAA,UES2HEA,XMPRMA2,
2F2*WEDS2HE,G2,CAPG,SMLG,XK)
CALL COMPUTE(NMAXG,GTEST,NUMDELE,DELETA,EPSSLONE,CAPG,SMLG,G2) )

C
C      C A L C U L A T E   T H E T A S
C
75 CONTINUE
ICHDR=3
THETAPR=THETA2(2)
CALL ABCDGS(NUMDELE,DELETA,XIBARA,DELXI,FA,VA,XMSTRA2,ICHDR,F1,
1XIBCPUA,XIBCPPA,RHOEROA,G1,THETA1,XIBCPZA,THETAA,UES2HEA,XMPRMA2,
2F2*WEDS2HE,G2,CAPG,SMLG,XK)
IF(XI1.EQ.XI0.AND.ITFRATE.EQ.1)NMAXTH=1+1*NMAXF
CALL COMPUTE(NMAXTH,THTEST,NUMDELE,DELETA,EPSSLONE,CAPG,SMLG,
1THETA2)

C
C      U P D A T E   T H E T A S
C
DO 76 I=1,NUMETA
76 THETAA(I)=(THETA1(I)+THETA2(I))/2.
ITERATE=ITERATE+1
IF(ITERATE.EQ.2)GO TO 37

C
C      C O N V E R G E N C E   C R I T E R I A
C
IF(ARS((F2(2)-FPREV)/FPREV).LE.EPSSLONW)GO TO 90
GO TO 37
90 IF(G1(2).EQ.0.)GO TO 100
IF(ABS((G2(2)-GPREV)/GPREV).LE.EPSSLONW)GO TO 100
GO TO 37
100 IF(ABS((THETA2(2)-THETAPR)/THETAPR).LE.EPSSLONW)GO TO 110
GO TO 37
110 CONTINUE
C
C      O U T P U T
C
C
C      C O M P U T E   R H O E R H O S
C
DO 80 I=1,NUMETA
RHOERO2(I)=((1.-ZETAW2)*THETA2(I)+ZETAW2-UES2HE2**2*F2(I)**2-
1WEDS2HE**2*G2(I)**2)/(1.-UES2HE2**2-WEDS2HE**2)
RHORHOE(I)=1./RHOERO2(I)
IF(GEE2.NE.0.)XMMEG(I)=G2(I)*SQRT(RHORHOE(I))
XMMEF(I)=F2(I)*SQRT(RHORHOE(I))
80 CONTINUE
FNCRER1=RHOERO2(I)
DO 102 I=2,NUMETA
FNCRER2=RHOERO2(I)
SUMRER(I)=SUMRER(I-1)+(FNCRER2+FNCRER1)/2.*DELETA(I-1)
FNCRER1=FNCRER2
YDL(I)=(2.*X12)**XNRAR *1./RERS2*SUMRER(I)/(CAPRS*R2**J*UES2HE2)
102 CONTINUE
CALL FTLUP(X12,XXL,1,NUMX,X1TAB,XL)
IF(XLPR(2).EQ.0.)GO TO 116
IF(XXL.GE.XLPR(NPRINT))GO TO 103

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IF(ICOUNT.EQ.1)GO TO 103
GO TO 115
116 IF(NPRINT.EQ.NSTEPS)GO TO 101
NPRINT=NPRINT+1
IF(ICOUNT.EQ.1)GO TO 103
GO TO 115
101 NPRINT=1
103 CONTINUE
DO 91 I=1,NUMETA
ZETA(I)=THETA2(I)*(1.-ZETAW2)+ZETAW2
91 CONTINUE
WRITE(6,111)XI2,XXL
111 FORMAT(1H0,3HXI=,E20.8,4X,4HX/L=,E20.8)
MAX=NMAXF+10
GO TO (117,118,119)IVEG
117 WRITE(6,112)
112 FORMAT(1H0,7X,3HETA,11X,3HY/L,12X,1HF,13X,1HV,11X,5HTHETA,9X,
14HZETA,8X,8HRHO/RHOE,8X,5HM/MEF)
DO 113 I=1,MAX
WRITE(6,114)I,ETA(I),YDL(I),F2(I),VA(I),THETA2(I),ZETA(I),
1RHORHOE(I),XMMEF(I)
113 CONTINUE
GO TO 115
114 FORMAT(14.9E14.4)
118 WRITE(6,141)
141 FORMAT(1H0,7X,3HETA,11X,3HY/L,12X,1HF,11X,6HEPSDMU,8X,5HTHETA,
19X,4HZETA,8X,8HRHO/RHOE,8X,5HM/MEF)
DO 142 I=1,MAX
WRITE(6,114)I,ETA(I),YDL(I),F2(I),EPSDMU(I),THETA2(I),ZETA(I),
1RHORHOE(I),XMMEF(I)
142 CONTINUE
GO TO 115
119 WRITE(6,143)
143 FORMAT(1H0,7X,3HETA,11X,3HY/L,12X,1HF,13X,1HG,11X,5HTHETA,9X,
14HZETA,8X,8HRHO/RHOE,8X,5HXMBAR,11X,5HXMMEF)
DO 144 I=1,MAX
WRITE(6,114)I,ETA(I),YDL(I),F2(I),G2(I),THETA2(I),ZETA(I),
1RHORHOE(I),XMBAR2(I),XMMEF(I)
144 CONTINUE
115 CONTINUE

C      U P D A T E   V A L U E S
C
XIBAR2=(2.*XI2)**(2.*XNBAR)
DO 120 I=1,NUMETA
F1(I)=F2(I)
G1(I)=G2(I)
THETA1(I)=THETA2(I)
RHOERO1(I)=RHOERO2(I)
120 CONTINUE
UES2HE1=UES2HE2
DO 106 I=1,NUMETA
HDCAPHE(I)=(1.-ZETAW2)*THETA1(I)+ZETAW2-UES2HE1**2*F1(I)**2
1-WEDS2HE**2*G1(I)**2
106 CONTINUE
CALL CALCM(ZETAW2,           UES2HE1,F1,WEDS2HE,           ZETA,
1               SHE,PR,           XMBAR1,XMCIRC1,XMSTAR1,XMPRIM1,PHIR,
2HSHE,           RERS2,X12,XNBAR ,R2,J,
3DELETA,RURDRUS,           CAPRS,SUMRER,AP,BP,CP,           ABTAB,FCFTAB,NFCFAB.

```

```

4NMAXG,G1 )
IF(ICOUNT.EQ.1)GO TO 104
IF(XLPR(2).EQ.0.)GO TO 145
IF(XXL .GE.XLPR(NPRINT))GO TO 105
GO TO 123
105 NPRINT=NPRINT+1
GO TO 104
145 IF(NPRINT.NE.1)GO TO 123
104 CONTINUE
XINB=(2.*X12)**XNBAR
CFF=PHIR(1)/RERS2*2.*R2**J/XINB*(F2(2)-F2(1))/DELETA(1)*RURDRUS
IF(GEE2.NE.0.)CFG=PHIR(1)/RERS2*2.*R2**J/XINB*(G2(2)-G2(1))/DELETA
1(1)*UES2HE1/WEDS2HE*RURDRUS
IF(GEE2.NE.0.)TAFDTAG=CFF/CFG*(UES2HE1/WEDS2HE)**2
SUMF=0.0
FUNCF1=F2(1)*(1.-F2(1))
DO 92 I=2,NMAXF
FUNCF2=F2(I)*(1.-F2(I))
SUMF=SUMF+(FUNCF2+FUNCF1)/2.*(ETA(I)-ETA(I-1))
FUNCF1=FUNCF2
92 CONTINUE
RCURJ=RERS2*CAPRS*UES2HE2*R2**J
TSDLF=XINB/RCURJ*SUMF
IF(GEE2.EQ.0.)GO TO 150
SUMG=0.0
FUNC G1=G2(1)*(1.-G2(1))
DO 94 I=2,NMAXG
FUNC G2=G2(I)*(1.-G2(I))
SUMG=SUMG+(FUNC G2+FUNC G1)/2.*(ETA(I)-ETA(I-1))
FUNC G1=FUNC G2
94 CONTINUE
TSDLG=XINB/RCURJ*SUMG
150 CONTINUE
SMDEL=0.0
SMTHE=0.0
FNCDEL1=1.-F2(1)
FNCTHE1=F2(1)*(1.-F2(1))
DO 56 I=2,NMAXF
FNCDEL2=1.-F2(I)
FNCTHE2=F2(I)*(1.-F2(I))
SMDEL=SMDEL+(FNCDEL2+FNCDEL1)/2.*(YDL(I)-YDL(I-1))
SMTHE=SMTHE+(FNCTHE2+FNCTHE1)/2.*(YDL(I)-YDL(I-1))
FNCDEL1=FNCDEL2
FNCTHE1=FNCTHE2
56 CONTINUE
HISF=SMDEL/SMTHE
IF(GEE2.EQ.0.)GO TO 151
SMDEL=0.0
SMTHE=0.0
FNCDEL1=1.-G2(1)
FNCTHE1=G2(1)*(1.-G2(1))
DO 57 I=2,NMAXG
FNCDEL2=1.-G2(I)
FNCTHE2=G2(I)*(1.-G2(I))
SMDEL=SMDEL+(FNCDEL2+FNCDEL1)/2.*(YDL(I)-YDL(I-1))
SMTHE=SMTHE+(FNCTHE2+FNCTHE1)/2.*(YDL(I)-YDL(I-1))
FNCDFL1=FNCDEL2
FNCTHE1=FNCTHE2

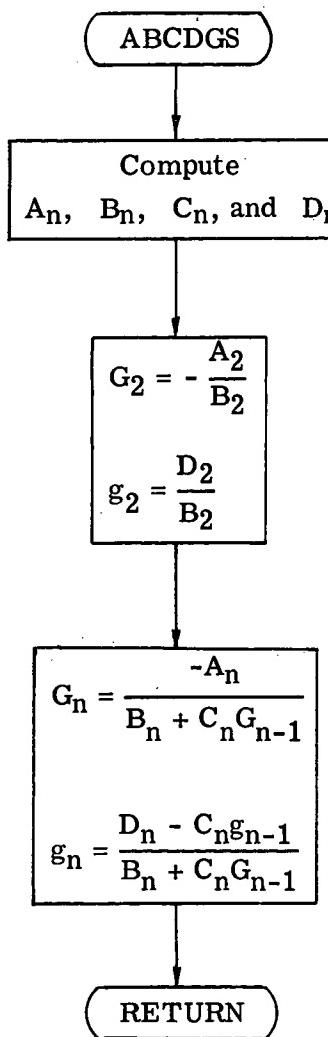
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57 CONTINUE
HISG=SMDEL/SMTHE
151 CONTINUE
SUMFRR=0.0
FNCFRR1=1.-F2(1)*RHORHOE(1)
DO 93 I=2,NMAXF
FNCFRR2=1.-F2(I)*RHORHOE(I)
SUMFRR=SUMFRR+(FNCFRR2+FNCFRR1)/2.*(YDL(I)-YDL(I-1))
FNCFRR1=FNCFRR2
93 CONTINUE
DSDLF=SUMFRR
IF(GEE2.EQ.0.)GO TO 96
SUMGRR=0.0
FNCGRR1=1.-G2(1)*RHORHOE(1)
DO 97 I=2,NMAXG
FNCGRR2=1.-G2(I)*RHORHOE(I)
SUMGRR=SUMGRR+(FNCGRR2+FNCGRR1)/2.*(YDL(I)-YDL(I-1))
FNCGRR1=FNCGRR2
97 CONTINUE
DSDLG=SUMGRR
96 CONTINUE
REX=CAPRS*XXL*UES2HE2*RERS2**2/RURDRUS
RDXL=REX/XXL
RETSF=RDXL*TSDLF
IF(GEE2.NE.0.)RETSG=RDXL*TSDLG
1*WEDS2HE/UES2HE2
REDSF=RDXL*DSDLF
IF(GEE2.NE.0.)REDSG=RDXL*DSDLG
1*WEDS2HE/UES2HE2
HEDCPHE=1.-UES2HE2**2-WEDS2HE**2
QBAR=PHIR(1)*RURDRUS*R2**J*UES2HE2*CAPRS/(PR*XINB)*(ZETA(2)
1 -ZETAW2)/DELETA(1)
ST=QBAR/(RERS2*CAPRS*UES2HE2*(FRO*(1.-HEDCPHE)+HEDCPHE-ZETAW2))
WRITE(6,121)
121 FORMAT(1H0,7X,3HCFF,10X,9HTHETA*/LF,7X,9HDELT A*/LF,9X,3HREX,11X,
19HRETHETA*F,7X,9HREDFLT A*F,9X,4HQBAR,13X,2HST)
WRITE(6,122)CFF,TSDLF,DSDLF,REX,RETSF,REDSF,QBAR,ST
122 FORMAT(8E16.4)
WRITE(6,124)
124 FORMAT(1H0,7X,4HHI*F)
WRITE(6,122)HISF
IF(GEE2.EQ.0.)GO TO 98
WRITE(6,131)
131 FORMAT(1H0,7X,3HCFG,10X,9HTAUF/TAUG,6X,9HTHETA*/LG,7X,
19HDELT A*/LG,7X,9HRETHETA*G,7X,9HREDELT A*G,9X,4HHI*G)
WRITE(6,122)CFG,TAFTDTAG,TSDLG,DSDLG,RETSG,REDSG,HISG
98 CONTINUE
ICOUNT=ICOUNT+1
123 CONTINUE
IF(XISTOP.LE.XI2)GO TO 130
XI1=XI2
XIBAR1=XIBAR2
CAPU1=CAPU2
CAPP1=CAPP2
CAPZ1=CAPZ2
GO TO 40
130 CONTINUE
STOP
END

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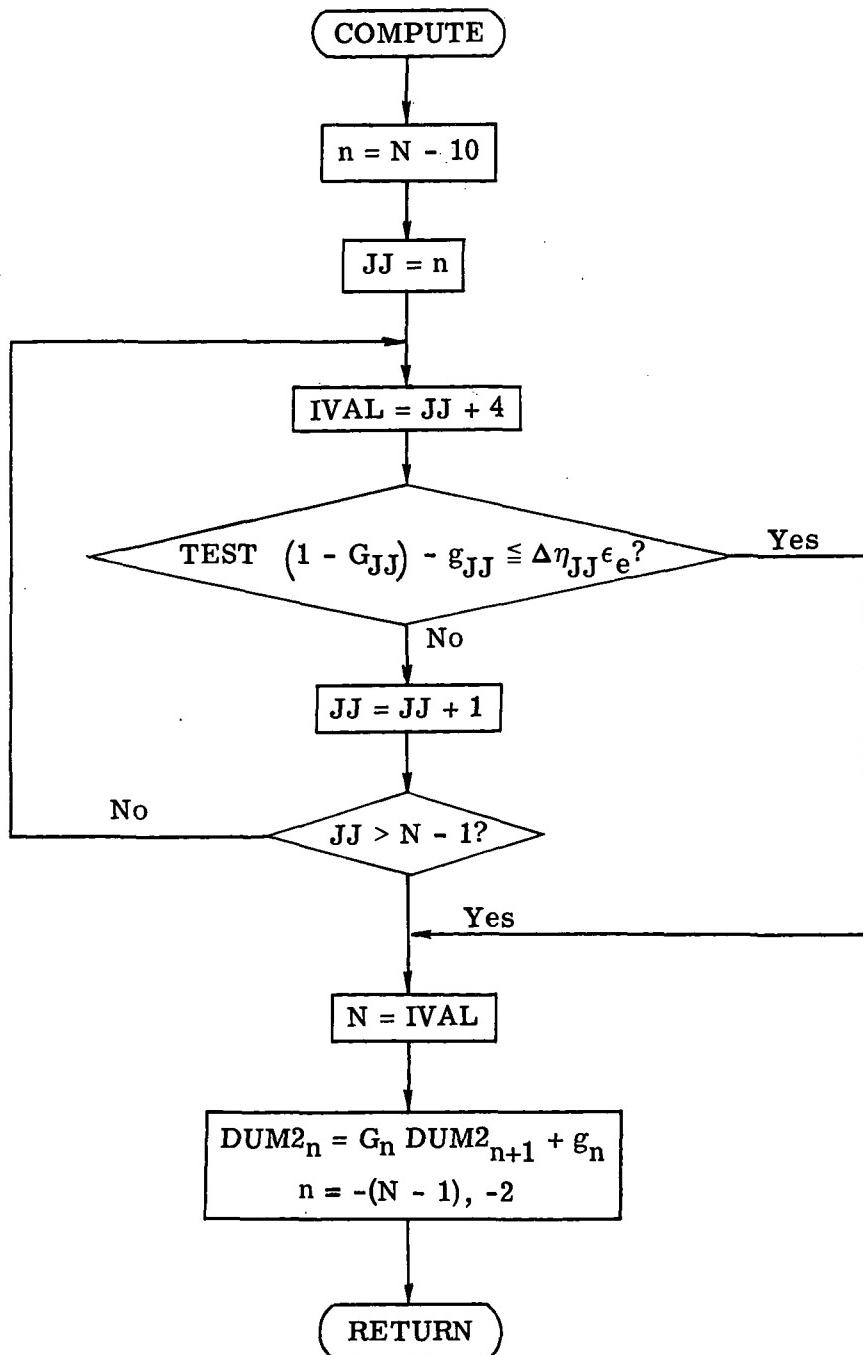
ABCDGS.- Subroutine ABCDGS computes coefficients G_n and g_n for the boundary layer equations by using recursion formulas (32). The flow diagram for subroutine ABCDGS is as follows: (In this flow diagram, A, B, C, D, G, and g are used as general notation for such coefficients as \bar{A} , A^* , \hat{A} , etc.)



The program listing for subroutine ABCDGS is as follows:

```
SUBROUTINE ABCDGS (NUMDELE, DELETA, XIBARA, DELXI, FA, VA, CAPMA2, ICHD,
1F1, XIBCPUA, XIBCPPA, RHOEROA, G1, THETA1, XIBCPZA, THETAA, UES2HEA,
2XMPRMA2, F2, WEDS2HF, G2, CAFG, SMLG, XK)
DIMENSION DELETA(350), F1(350), F2(350), FA(350), G1(350), G2(350),
1THETA1(350), THETA2(350), THETAA(350), VA(350), CAPMA2(350), CAPA(350),
2CAPB(350), CAPC(350), CAPD(350), RHOEROA(350), XMPRMA2(350), CAPG(350),
3SMLG(350)
FCT1=2.*XK/(1.+XK)
FCT2=2./ (1.+XK)
DO 10 I=2,NUMDELE
DELDEL2=2.*DELETA(I)*DELETA(I-1)
XIDELFA=XIBARA/DELXI*FA(I)
VADLDL2=VA(I)/(2.*(DELETA(I)+DELETA(I-1)))
TMDLDL2=(CAPMA2(I)*FCT2+CAPMA2(I-1)*FCT1)/DELDEL2
CAPA(I)=VADLDL2-CAPMA2(I)*FCT2/DELDEL2
CAPB(I)=XIDELFA+TMDLDL2
CAPC(I)=- (VADLDL2+CAPMA2(I-1)*FCT1/DELDEL2)
IF (ICHD.EQ.1) GO TO 20
IF (ICHD.EQ.2) GO TO 30
IF (ICHD.EQ.3) GO TO 40
20 CAPD(I)=-CAPA(I)*F1(I+1)+(XIDELFA-TMDLDL2)*F1(I)-CAPC(I)*F1(I-1)
1-XIBCPUA*FA(I)**2-XIBCPPA*RHOEROA(I)
GO TO 10
30 CAPD(I)=-CAPA(I)*G1(I+1)+(XIDELFA-TMDLDL2)*G1(I)-CAPC(I)*G1(I-1)
GO TO 10
40 CAPD(I)=-CAPA(I)*THETA1(I+1)+(XIDELFA-TMDLDL2)*THETA1(I)-CAPC(I)*
1THETA1(I-1)+XIBCPZA*FA(I)*(THETAA(I)-1.)-UES2HEA**2/DELDEL2*
2(XMPRMA2(I)*FCT2*(F1(I+1)**2+F2(I+1)**2-F1(I)**2-F2(I)**2)
3-XMPRMA2(I-1)*FCT1*(F1(I)**2+F2(I)**2-F1(I-1)**2-F2(I-1)**2)
4-WEDS2HE**2/DELDEL2*(XMPRMA2(I)*FCT2*(G1(I+1)**2+G2(I+1)**2
5-G1(I)**2-G2(I)**2)-XMPRMA2(I-1)*FCT1*(G1(I)**2+G2(I)**2-G1(I-1)
6**2-G2(I-1)**2))
10 CONTINUE
CAPG(2)=-CAPA(2)/CAPR(2)
SMLG(2)=CAPD(2)/CAPB(2)
DO 50 I=3,NUMDELE
BCG=CAPB(I)+CAPC(I)*CAPG(I-1)
CAPG(I)=-CAPA(I)/BCG
SMLG(I)=(CAPD(I)-CAPC(I)*SMLG(I-1))/BCG
50 CONTINUE
RETURN
END
```

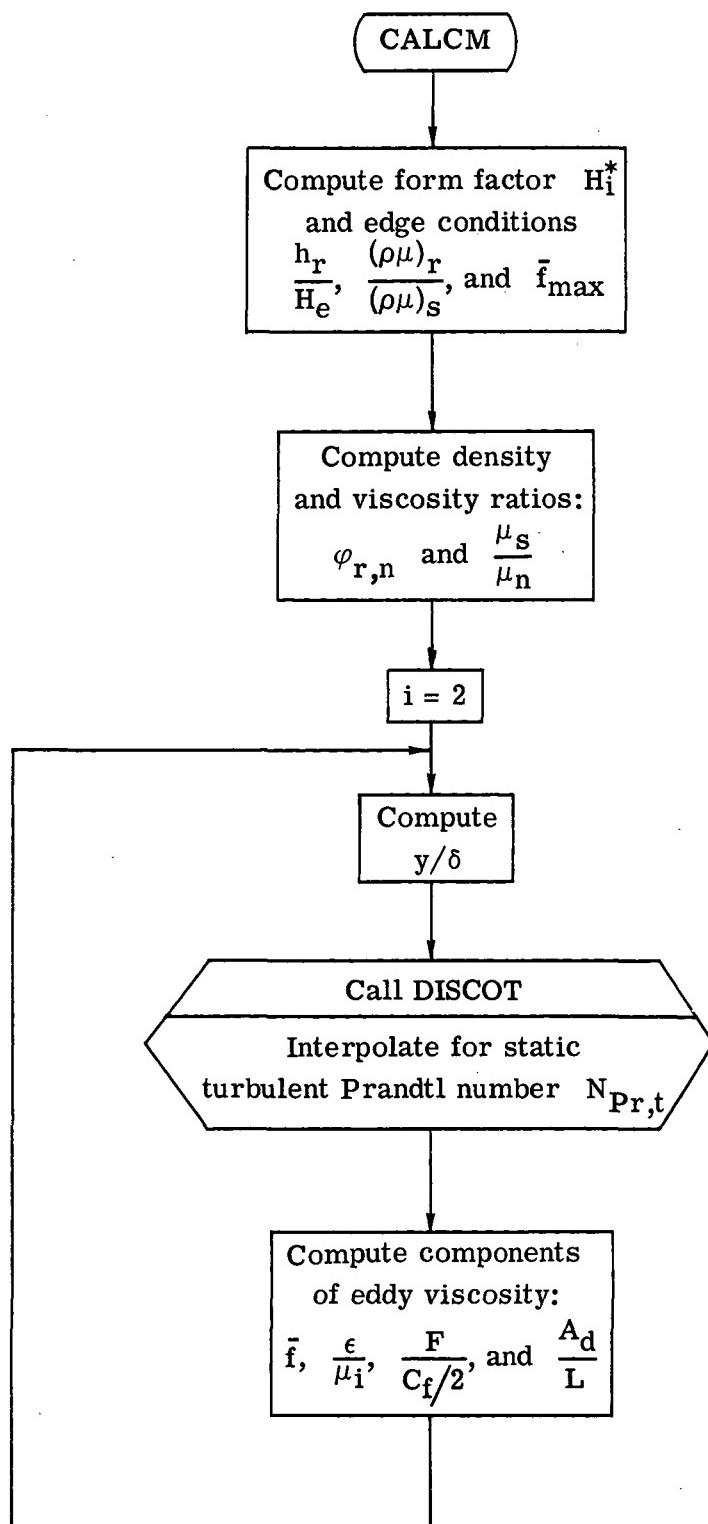
COMPUTE.- Subroutine COMPUTE determines the number of equations (or $\Delta\eta$ steps) to be used and computes new values for the dependent variables by using formula (31), (42), or (43). The flow diagram for subroutine COMPUTE is as follows: (In this flow diagram, G and g are used as general notation for such coefficients as \bar{G} , \bar{g} , G^* , etc.)

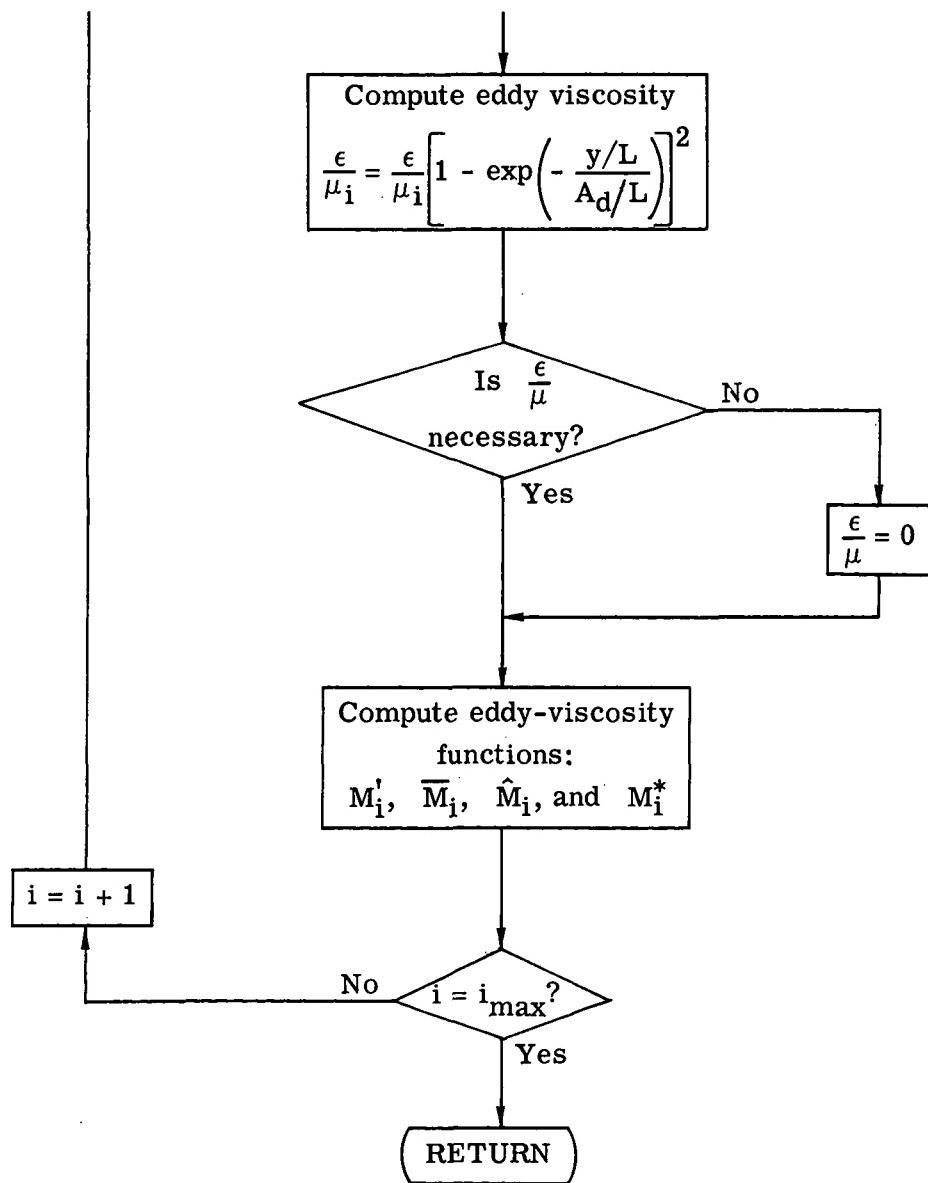


The program listing for subroutine COMPUTE is as follows:

```
SUBROUTINE COMPUTE(NMAX,TEST,NUMDELE, DELETA,EPSSLONE,CAPG,SMLG,
1 DUM2 )
DIMENSION CAPG(350),SMLG(350),DUM2(350),DELETA(350)
C
C      T E S T - F I N D I N G   E D G E
C
KMAX=NMAX-10
DO 10 JJ=KMAX,NUMDEL
TSTVAL=EPSSLONE*DELETA(JJ)
CKVAL=TEST*(1.0-CAPG(JJ))-SMLG(JJ)
IVAL=JJ+4
IF(ABS(CKVAL).LE.ABS(TSTVAL))GO TO 20
10 CONTINUE
20 NMAX=IVAL
C
C      C O M P U T E   D U M 2 S
C
21 CONTINUE
NBACK=-(NMAX-1)
M2=-2
DO 30 NF=NBACK,M2
KF=IABS(NF)
DUM2(KF)=CAPG(KF)*DUM2(KF+1)+SMLG(KF)
30 CONTINUE
RETURN
END
```

CALCM.- Subroutine CALCM computes the viscosity functions. The flow diagram for this subroutine is as follows:





The program listing for subroutine CALCM is as follows:

```
SUBROUTINE CALCM(ZETAW,UEDS2HE,F2,WEDS2HE,ZETAE,
1                   SHE,PR,XMBAR,XMCIRC,XMSTAR,XMPRIM,PHIR,HSHE,
2                   RERS,XI,XNBAR,R,J,DELETA,
3RURDRUS,CAPRS,SUMRER,AP,BP,CP,ABTAB,FCFTAB,NFCFAB,NMAXG,
4G2)
DIMENSION F2(350),G2(350),XMBAR(350),XMCIRC(350),
1XMSTAR(350),XMPRIM(350),PHIR(350),DELETA(350),SUMRER(350),
2ABTAB(20),FCFTAB(20),EMUSDMU(350)
COMMON/EPSDMU/EPSDMU(350)
COMMON/TABLE1/RHOERO2(350)
COMMON/TABLE2/YDL(350),RHORHOE(350)
COMMON/THREE/NUMETA,NMAXF
COMMON/FEB12/VWA
COMMON/FEB11/PRTTAB(20),YDDPRT(20),NYP
COMMON/HDCAPHE/HDCAPHE(350),GEE2
COMMON/MUUSE/IUSEEMU,MPWFMU
COMMON/FBAR/FBARTAB(20),IFBLU,YDDFB(20),NFBY
COMMON/IWLDMP/IWLDMP
HRDCPHE=ZETAE-UEDS2HE**2-WEDS2HE**2
RURDRUS=SQRT(HRDCPHE/HSHE)*(HSHE+SHE)/(HRDCPHE+SHE)
1*RERS*HRDCPHE
2/HSHE
RUSDRUR=1./RURDRUS
SMRER=0.0
FNCRER1=RHOERO2(1)
IF(GEE2.NE.0.)GO TO 24
DO 25 I=2,NMAXF
FNCRER2=RHOERO2(I)
SMRER=SMRER+(FNCRER2+FNCRER1)/2.*DELETA(I-1)
IF(F2(I).GE..995)GO TO 23
FNCRER1=FNCRER2
25 CONTINUE
GO TO 23
24 CONTINUE
DO 22 I=2,NMAXG
FNCRER2=RHOERO2(I)
SMRER=SMRER+(FNCRER2+FNCRER1)/2.*DELETA(I-1)
IF(G2(I).GE..995)GO TO 23
FNCRER1=FNCRER2
22 CONTINUE
23 CONTINUE
SMDEL=0.0
SMTHE=0.0
IF(GEE2.NE.0.)GO TO 26
FNCDEL1=1.-F2(1)
FNCTHE1=F2(1)*(1.-F2(1))
DO 27 I=2,NMAXF
FNCDEL2=1.-F2(I)
FNCTHE2=F2(I)*(1.-F2(I))
SMDEL=SMDEL+(FNCDEL2+FNCDEL1)/2.*(YDL(I)-YDL(I-1))
SMTHE=SMTHE+(FNCTHE2+FNCTHE1)/2.*(YDL(I)-YDL(I-1))
FNCDEL1=FNCDEL2
FNCTHE1=FNCTHE2
27 CONTINUE
GO TO 28
```

```

26 CONTINUE
FNCDEL1=1.-G2(1)
FNCTHE1=G2(1)*(1.-G2(1))
DO 11 I=2,NMAXG
FNCDFL2=1.-G2(I)
FNCTHE2=G2(I)*(1.-G2(I))
SMDEL=SMDEL+(FNCDEL2+FNCDEL1)/2.*(YDL(I)-YDL(I-1))
SMTHE=SMTHE+(FNCTHE2+FNCTHE1)/2.*(YDL(I)-YDL(I-1))
FNCDEL1=FNCDEL2
FNCTHE1=FNCTHE2
11 CONTINUE
28 CONTINUE
HIS=SMDEL/SMTHE
FBARMAX=AP+BP*HIS+CP*HIS*HIS
DO 12 I=1,NUMETA
PHIR(I)=SQRT(HDCAPHE(I)/HRDCPHE)*(HRDCPHE+SHE)/(HDCAPHE(I)+SHE)
EMUSDMU(I)=RUSDRUR*RHORHOE(I)*RERS/PHIR(I)
12 CONTINUE
DO 10 I=1,NUMETA
YDD=SUMRER(I)/SMRER
CALL DISCOT(YDD,YDD,YDDPRT,PRTTAB,PRTTAB,-11,NYP,0,PRT)
IF(I.EQ.1)GO TO 32
IF(IFBLU.EQ.1)GO TO 13
IF(YDD.LE..1)FBAR=.4*YDD
IF(YDD.GT..1.AND.YDD.LE..3)FBAR=.04+((YDD-.1)/.2)*(FBARMAX-.04)
IF(YDD.GT..3)FBAR=FBARMAX
GO TO 14
13 CALL DISCOT(YDD,YDD,YDDFB,FBARTAB,FBARTAB,-11,NFBY,0,FBAR)
14 CONTINUE
IF(I.EQ.NUMETA)GO TO 32
IF(GEE2.NE.0.)GO TO 29
EPSDMU(I)=(2.*XI)**XNBAR/R**J* FBAR**2*EMUSDMU(I)*RHORHOE(I)**2*
1SMRER**2
1*ABS((F2(I+1)-F2(I-1))/(DELETA(I)+DELETA(I-1)))
GO TO 34
29 CONTINUE
EPSDMU(I)=(2.*XI)**XNBAR/R**J*FBAR**2*EMUSDMU(I)*RHORHOE(I)**2
1*SMRER**2
1*SQRT(ABS(((F2(I+1)-F2(I-1))/(DELETA(I)+DELETA(I-1)))*2
1+((G2(I+1)-G2(I-1))/(DELETA(I)+DELETA(I-1)))*2
1*(WEDS2HF/UEDS2HE)**2)
34 CONTINUE
IF(IWLDP.EQ.1.AND.I.NE.2)GO TO 15
IF(I.NE.2)GO TO 33
IF(GEE2.NE.0.)GO TO 40
FDCFD2=((RHORHOE(I)*VWA)/(PHIR(I)/RERS*(2.*R**J)/((2.*XI)**XNBAR)*
1F2(2)/DELETA(I)*RURDRUS))*2.
GO TO 41
40 FDCFD2=((RHORHOE(I)*VWA)/(PHIR(I)/RERS*(2.*R**J)/((2.*XI)**XNBAR)*
1SQRT((F2(2)/DELETA(I))**2+(G2(2)/DELETA(I))**2*(WEDS2HE/UEDS2HE)**2
22)*RURDRUS))*2.
2*UEDS2HE/SQRT(WEDS2HF**2+UEDS2HE**2)
41 CONTINUE
CALL FTLLUP(FDCFD2,AB,1,NFCFAB,FCFTAB,ABTAB)
IF(GEE2.NE.0.)GO TO 35
15 CONTINUE
IF(IWLDP.EQ.1)GO TO 16
ADL= AB*SQRT((2.*XI)**XNBAR)/(CAPRS*SQRT(EMUSDMU(I))*RHORHOE(I))

```

```

1*RFRS
1*UEDS2HE*SQRT(R**J)*SQRT(ABS(F2(2))/DELETA(1)))
GO TO 36
16 ADL=AB*SQRT((2.*XI)**XNBAR)*SQRT(EMUSDMU(1))/(CAPRS*EMUSDMU(1))
1*SQRT(RHORHOE(1))*SQRT(RHORHOE(1))*RERS
1*UEDS2HE*SQRT(R**J)*SQRT(ABS(F2(2))/DELETA(1)))
GO TO 36
35 CONTINUE
ADL= AB*SQRT((2.*XI)**XNBAR)/(CAPRS*SQRT(EMUSDMU(1))*RHORHOE(1))
1*RFRS
1*UFDS2HE*SQRT(R**J)
1*((F2(2)/DELETA(1))**2+(G2(2)/DELETA(1))**2*(WEDS2HE/UEDS2HE)**2)
1**.25)
36 CONTINUE
33 EPNSDMU(I)=EPNSDMU(I)*(1.-EXP(-YDL(I)/ADL))**2
51 IF(EPNSDMU(I).LT.0.)EPNSDMU(I)=0.
GO TO 31
32 EPNSDMU(I)=0.0
31 CONTINUE
IF(IUSEEMU.EQ.0)EPNSDMU(I)=0.
XMBAR(I)=PHIR(I)*(1.+EPNSDMU(I))
XMCIRC(I)=XMBAR(I)
XMSTAR(I)=PHIR(I)/PR*(1.+EPNSDMU(I)*PR/PRT)
IF(MPWFMU.EQ.1)GO TO 37
XMPRIM(I)=PHIR(I)/PR*(1.-PR)/(1.-ZETAW)
GO TO 10
37 XMPRIM(I)=PHIR(I)/PR*(1.-PR)/(1.-ZETAW)*(1.+EPNSDMU(I)*PR/PRT*(1.-P
1RT)
1/(1.-PR))
10 CONTINUE
RETURN
END

```

USAGE

The program is run on the Control Data 6000 series computer under the SCOPE 3.0 operating system. Minimum machine requirements are 75 000 octal locations of core storage. The time required to calculate a grid point is approximately 0.002 second per iteration. Each x-step typically uses three iterations. The restrictions and usual values for input quantities are given in their description.

Input Description

The FORTRAN NAMELIST capability is used for data input with NAM1 as the NAMELIST name. The maximum allowable dimension appears following the variable name.

NUMETA maximum number of steps in η direction, 350 maximum

NMAXF initial guess at number of η steps to outer edge of F profile,
generally out to $F = 0.999$

Note: The procedure to obtain NMAXF is to (a) estimate $\Delta\eta$, making
sure that there are at least four steps in linear portion of profile
near wall, (b) choose a K value, (c) estimate an η_e value,
and (d) solve equation (24) for $N = NMAXF$. Generally,
 $NUMETA = NMAXF + 50$ has been used successfully.

NMAXG initial guess at number of η steps to outer edge of g profile,
generally out to $g = 0.999$

DELETA value of $\Delta\eta$ nearest wall (that is, $\Delta\eta_1$); there must be at least four
steps in linear portion of profile near wall

XK $\Delta\eta_n/\Delta\eta_{n-1}$ ratio, generally taken as 1.02

XI0 value of ξ at input station

DELXI0 initial $\Delta\xi$, generally $(10^{-3})(\xi_0)$

XTEST value of ξ where $\Delta\xi$ is increased by a factor of 10

XSTOP value of ξ where solution is to be terminated

| | | |
|---------------------|-------|--|
| FTAB | (100) | input u/u_e profile (values must correspond to YL table); there must be at least four points in linear region near wall |
| ETATAB | (100) | input η table, used when input profiles are known as $f(\eta)$ rather than $f(y/L)$ (it can be omitted as input if not required) |
| VWTAB | (75) | v_w/u_e for axisymmetric or two-dimensional flow, v_w/w_e for swept-cylinder problem (values must correspond to XL table) |
| EPSL \emptyset NE | | accuracy criteria for $\partial(F,g,\theta)/\partial\eta$ at outer edge of profile, generally set equal to 0.05 for turbulent flows and 0.0001 for laminar flows |
| EPSL \emptyset NW | | convergence criteria for iterations on F , g , and θ profiles; allowable percent change in wall slope between iterations, 0.01 generally used |
| GTAB | (100) | input w/w_e profile (values must correspond to YL table and must be zero if not used) |
| UEDSTAB | (75) | $u_e/\sqrt{2H_e}$ table (values must correspond to XL table) |
| WEDS2HE | | $w_e/\sqrt{2H_e}$ |
| PR | | molecular Prandtl number |
| ZETWTAB | (75) | ζ_w table (values must correspond to XL table) |
| XNBAR | | \bar{n} , generally taken as 0.5 for laminar flows and 0.8 for turbulent flows |
| RERSTAB | (75) | ρ_e/ρ_s table (values must correspond to XL table) |
| CAPRS | | reference Reynolds number, $\rho_s \sqrt{2H_e} L / \mu_s$; the subscript "s" must be taken as isentropic stagnation conditions except for swept-cylinder problems in which "s" is taken as stagnation-line values |
| RTAB | (75) | r/L table (values must correspond to XL table) |

| | |
|---------------|---|
| J | body shape index ($j = 0$ for two-dimensional flows; $j = 1$ for axisymmetric flows) |
| RHOTAB (100) | input ρ_e/ρ table, can be used instead of input ζ profile, values must correspond to YL table (can be omitted as input if not required) |
| SHE | Sutherland's constant, $S/H_e = 202/T_s$, where T_s is in $^{\circ}$ R for air |
| ZETATAB (100) | input ζ profile, values must correspond to YL table (can be omitted as input if not required) |
| ITHETA | code for input temperature profile (ITHETA = 1 if RHOTAB used; ITHETA = 2 if ZETATAB used) |
| HSHE | yaw parameter, $1 - \frac{w_e^2}{2H_e}$ |
| XL (75) | table of x/L values, first entry must equal x_0 value |
| NUMX | number of values in XL table, 75 maximum |
| YL (100) | table of y/L values for initial profiles, first entry must equal zero |
| NUMY | number of values in YL table, 100 maximum |
| X0 | initial value of x/L , must not equal zero |
| ØL | reference length, given in feet |
| DUDXTAB (75) | $d(u_e/\sqrt{2H_e})/d(x/L)$ table, values must correspond to XL table |
| DZDXTAB (75) | $d\xi_w/d(x/L)$ table, values must correspond to XL table |
| FR0 | temperature recovery factor, generally 0.85 for laminar flows and 0.89 for turbulent flows; used to compute Stanton number |
| NSTEPS | number of ξ steps between profile printouts (can be omitted as input if XLPR table is used to designate printout) |

AP
BP
CP}

constants A, B, and C for \bar{f}_{\max} (see eq. (56a))

| | | | |
|---------|------|--|---|
| ABTAB | (20) | table of A_b values in Van Driest damping function (eq. (59)), input as $f(FCFTAB)$ | Generally use values from sample case shown |
| FCFTAB | (20) | $\frac{\rho_w v_w}{\rho_e u_e} \frac{2}{C_f}$ values corresponding to ABTAB values | |
| NFCFAB | | number of ABTAB values in table, 20 maximum | |
| PRTTAB | (20) | turbulent Prandtl number table, input as $f(y/\delta)$ | |
| YDDPRT | (20) | y/δ values corresponding to PRTTAB, values must start at zero | |
| NYP | | number of values in PRTTAB table, 20 maximum | |
| XLPR | (30) | x/L values where profile printout is required (can be omitted as input if NSTEPS is used) | |
| IVEG | | = 1 for printout of V profile = 2 for ϵ/μ profile = 3 for g profile | |
| INIT | | = 0 for $\Delta\xi_{initial} = DELXI0$ = 1 for small $\Delta\xi_{initial}$; after 30 ξ steps, $\Delta\xi$ is set back to $DELXI0$ (used to smooth input profile) | |
| IUSEEMU | | = 0 for laminar ($\epsilon/\mu \equiv 0$) solution = 1 for turbulent solution | |
| MPWEMU | | = 0 for $PRTTAB = N_{Pr,T}$ values = 1 for $PRTTAB = N_{Pr,t}$ values | |
| FBARTAB | (20) | \bar{f} values corresponding to YDDFB values | |
| IFBLU | | = 0 for computing \bar{f} = 1 for table lookup of \bar{f} | |

YDDFB (20) y/δ values corresponding to FBARTAB

NFBY number of FBARTAB values in table

IWLDMF
= 0 for using wall properties in wall damping function
= 1 for using local properties in wall damping function
(should be set equal to 0 for swept-leading-edge situation)

Output Description

The output of program D2630 consists of printing only. The main program prints the NAMELIST input and the output described below. If IVEG = 2 in the input, the ϵ/μ profile will be printed as in the sample output. For IVEG = 1, the V profile is printed in the place of EPSDMU, whereas IVEG = 3 prints the g profile. The frequency of the output is controlled by one of two input quantities. If a table of x/L values is given for XLPR, printout will appear for the computation step nearest each value. Otherwise, NSTEPS is used and denotes the number of ξ steps between profile printouts.

XI ξ

X/L x/L

ETA η

Y/L y/L

F F

EPSDMU ϵ/μ (for IVEG = 2); V (for IVEG = 1); G (for IVEG = 3)

THETA θ

ZETA ζ

RHO/RHOE ρ/ρ_e

M/MEF $(M/M_e)_F$

CFF wall-skin-friction coefficient for chordwise profile

| | |
|-----------|---|
| THETA*/LF | $(\theta^*/L)_F$ |
| DELTA*/LF | $(\delta^*/L)_F$ |
| REX | $\frac{\rho_e u e^x}{\mu_e}$ |
| RETHETA*F | $\frac{\rho_e u e^{\theta^*}_F}{\mu_e}$ |
| REDELTA*F | $\frac{\rho_e u e^{\delta^*}_F}{\mu_e}$ |
| QBAR | heat-transfer parameter $\dot{q}_w L / (\mu_s H_e)$ |
| ST | Stanton number, computed using input FR0 |
| HI*F | $(H_i^*)_F$ |

If a spanwise velocity profile "g" is used, the following output also appears:

| | |
|-----------|---|
| CFG | wall-skin-friction coefficient for spanwise profile |
| TAUF/TAUG | ratio of chordwise to spanwise shear at wall |
| THETA*/LG | $(\theta^*/L)_g$ |
| DELTA*/LG | $(\delta^*/L)_g$ |
| RETHETA*G | $\frac{\rho_e u e^{\theta^*}_g}{\mu_e}$ |
| REDELTA*G | $\frac{\rho_e u e^{\delta^*}_g}{\mu_e}$ |
| HI*G | $(H_i^*)_g$ |

Sample Case

The listing of the input data for the McLafferty-Barber Mach 3 adverse pressure gradient flow sample case, as described in reference 16, is as follows:

```
$NAM1  
NUMETA=230,  
NMAXF=100,  
NMAXG=1,  
DFLETA=2.E-2,  
XK=1.02,  
XI0=1.6E6,  
DELXI0=.15E3,  
X1TEST=1.E15,  
X1STOP=1.63705E6,  
FTAB=0.,.0858,.1716,.2574,.3432,.41,.5,.58,.627,.664,.683,.718,.743,  
.789,.826,.872,.912,.945,.964,.978,.988,.994,.999,.9995,.9997,.9999,1.,  
VWTAB=B*0.,  
EPSLONE=.5E-3,  
EPSLONW=.01,  
GTAB=27*0.,  
UEDSTAB=.8015,.790,.775,.754,.74,.701,.663,.625,  
WFDS2HE=0.,  
PR=.7,  
ZFTWTAB=8*.93,  
XNRAR=.5,  
RERSTAB=.0762,.0872,.101,.1218,.1377,.1841,.2352,.290,  
CAPRS=1.08E6,
```

RTAB=8*1.,
J=0.,
SHE=.331.,
ZETATAB=.93,,.936,,.942,,.948,,.954,,.9587,,.965,,.9706,,.9739,,.9765,,.9778,,
.9802,,.9820,,.9852,,.9878,,.9910,,.9938,,.9962,,.9975,,.9985,,.9992,,.9996,,.99993,,
.99996,,.99998,,.99999,1.,
ITHETA=2.,
HSHE=1.,
XL=.001,.3,.63,1,1.25,1.88,2.61,3.14.,
NUMX=8.,
YL=0.,.001,.002,.003,.004,.006,.01,.015,.020,.025,.030,.040,.050,.07,
.09,.12,.15,.18,.2,.22,.24,.26,.28,.3,.32,.34,1.,
NUMY=27.,
X0=.001.,
OL=.0834.,
DUDXTAB=-.0295,-.0415,-.05,-.0575,-.06,-.0615,-.06,-.058,
DZDXTAB=8*0.,
FR0=.89.,
NSTEPS=1.,
AP=.1.,
BP=0.,
CP=0.,
ABTAB(1)=60.,26.,18,5,14,8,12,5,11,2,10,9,1,6,3,8,3,,
FCFTAB(1)=-1.,0,1,2,3,4,5,6,12,24,100.,
NFCFAB=11.,
PRTTAB=4*9.,
YDDPRT=0.,5,1,15.,

NYP=4,

XLPR=3.63+1.25+1.88+2.51+3.14,

TVEG=2,

INIT=0,

IUSFFMU=1,

MPWEMU=1,

IFRLU=0,

IWLDMU=0,

\$

The sample output for this case is as follows:

| XI= | 1.60900000E+06 | X/L= | 3.02740161E-01 | ETA | Y/L | F | EPSDMU | THETA | ZETA | RHO/RHOE | M/MEF |
|------------|----------------|------------|----------------|------------|------------|------------|------------|------------|------------|------------|------------|
| 1 | 0. | 0. | 0. | 0. | 0. | 0. | 0. | 0. | 9.3000E-01 | 4.0441E-01 | 0. |
| 2 | 2.0000E-02 | 1.1904E-03 | 5.6497E-02 | 1.0750E-03 | 1.2946E-02 | 9.3091E-01 | 4.0488E-01 | 9.3091E-01 | 4.0488E-01 | 3.5949E-02 | 3.5949E-02 |
| 3 | 4.0400E-02 | 2.4009E-03 | 1.1945E-01 | 1.8160E-02 | 4.6906E-02 | 9.3328E-01 | 4.0686E-01 | 9.3328E-01 | 4.0686E-01 | 7.6192E-02 | 7.6192E-02 |
| 4 | 6.1208E-02 | 3.6268E-03 | 1.8632E-01 | 9.1657E-02 | 1.0450E-01 | 9.3732E-01 | 4.1074E-01 | 4.1074E-01 | 4.1074E-01 | 1.1941E-01 | 1.1941E-01 |
| 5 | 8.2432E-02 | 4.8626E-03 | 2.5176E-01 | 2.6789E-01 | 1.7880E-01 | 9.4252E-01 | 4.1651E-01 | 4.1651E-01 | 4.1651E-01 | 1.6248E-01 | 1.6248E-01 |
| 6 | 1.0408E-01 | 6.1038E-03 | 3.1059E-01 | 5.6683E-01 | 2.5572E-01 | 9.4790E-01 | 4.2367E-01 | 4.2367E-01 | 4.2367E-01 | 2.0216E-01 | 2.0216E-01 |
| 7 | 1.2616E-01 | 7.3637E-03 | 3.6076E-01 | 9.8223E-01 | 3.2488E-01 | 9.5274E-01 | 4.3153E-01 | 4.3153E-01 | 4.3153E-01 | 2.3699E-01 | 2.3699E-01 |
| 8 | 1.4869E-01 | 8.5932E-03 | 4.0276E-01 | 1.5030E+00 | 3.8302E-01 | 9.5681E-01 | 4.3957E-01 | 4.3957E-01 | 4.3957E-01 | 2.67C3E-01 | 2.67C3E-01 |
| 9 | 1.7166E-01 | 9.8408E-03 | 4.3796E-01 | 2.1211E+00 | 4.3073E-01 | 9.6015E-01 | 4.4748E-01 | 4.4748E-01 | 4.4748E-01 | 2.9297E-01 | 2.9297E-01 |
| 10 | 1.9509E-01 | 1.1091E-02 | 4.6771E-01 | 2.8311E+00 | 4.6980E-01 | 9.6289E-01 | 4.5510E-01 | 4.5510E-01 | 4.5510E-01 | 3.1552E-01 | 3.1552E-01 |
| . | . | . | . | . | . | . | . | . | . | . | . |
| . | . | . | . | . | . | . | . | . | . | . | . |
| . | . | . | . | . | . | . | . | . | . | . | . |
| 115 | 8.5592E+00 | 2.7093E-01 | 9.9940E-01 | 3.0451E+00 | 9.9948E-01 | 9.9996E-01 | 9.9811E-01 | 9.9846E-01 | 9.9846E-01 | 9.9846E-01 | 9.9846E-01 |
| 116 | 8.7503E+00 | 2.7554E-01 | 9.9950E-01 | 1.7780E+00 | 9.9951E-01 | 9.9997E-01 | 9.9842E-01 | 9.9871E-01 | 9.9871E-01 | 9.9871E-01 | 9.9871E-01 |
| 117 | 8.9453E+00 | 2.8025E-01 | 9.9956E-01 | 1.3587E+00 | 9.9957E-01 | 9.9997E-01 | 9.9863E-01 | 9.9888E-01 | 9.9888E-01 | 9.9888E-01 | 9.9888E-01 |
| 118 | 9.1443E+00 | 2.8504E-01 | 9.9962E-01 | 1.2634E+00 | 9.9964E-01 | 9.9997E-01 | 9.9881E-01 | 9.9903E-01 | 9.9903E-01 | 9.9903E-01 | 9.9903E-01 |
| 119 | 9.3471E+00 | 2.8993E-01 | 9.9968E-01 | 1.2481E+00 | 9.9971E-01 | 9.9998E-01 | 9.9900E-01 | 9.9918E-01 | 9.9918E-01 | 9.9918E-01 | 9.9918E-01 |
| 120 | 9.5541E+00 | 2.9462E-01 | 9.9974E-01 | 1.2394E+00 | 9.9975E-01 | 9.9998E-01 | 9.9919E-01 | 9.9934E-01 | 9.9934E-01 | 9.9934E-01 | 9.9934E-01 |
| 121 | 9.7652E+00 | 3.0001E-01 | 9.9980E-01 | 1.2665E+00 | 9.9980E-01 | 9.9999E-01 | 9.9938E-01 | 9.9949E-01 | 9.9949E-01 | 9.9949E-01 | 9.9949E-01 |
| 122 | 9.9805E+00 | 3.0520E-01 | 9.9987E-01 | 8.5618E-01 | 9.9987E-01 | 9.9999E-01 | 9.9959E-01 | 9.9959E-01 | 9.9959E-01 | 9.9959E-01 | 9.9959E-01 |
| 123 | 1.0200E+01 | 3.1049E-01 | 9.9998E-01 | 2.4930E-01 | 9.9998E-01 | 9.9999E-01 | 9.9966E-01 | 9.9972E-01 | 9.9972E-01 | 9.9972E-01 | 9.9972E-01 |
| 124 | 1.0424E+01 | 3.1589E-01 | 9.9998E-01 | 4.8238E-02 | 9.9998E-01 | 9.9999E-01 | 9.9967E-01 | 9.9973E-01 | 9.9973E-01 | 9.9973E-01 | 9.9973E-01 |
| 125 | 1.0653E+01 | 3.2139E-01 | 9.9998E-01 | 5.9639E-02 | 9.9986E-01 | 9.9999E-01 | 9.9968E-01 | 9.9973E-01 | 9.9973E-01 | 9.9973E-01 | 9.9973E-01 |
| 126 | 1.0886E+01 | 3.2701E-01 | 9.9990E-01 | 8.9812E-02 | 9.9986E-01 | 9.9999E-01 | 9.9970E-01 | 9.9975E-01 | 9.9975E-01 | 9.9975E-01 | 9.9975E-01 |
| 127 | 1.1123E+01 | 3.3273E-01 | 9.9990E-01 | 4.6930E-02 | 9.9986E-01 | 9.9999E-01 | 9.9971E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 |
| 128 | 1.1366E+01 | 3.3857E-01 | 9.9991E-01 | 1.7555E-02 | 9.9986E-01 | 9.9999E-01 | 9.9971E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 |
| 129 | 1.1613E+01 | 3.4453E-01 | 9.9991E-01 | 1.7555E-02 | 9.9987E-01 | 9.9999E-01 | 9.9972E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 | 9.9976E-01 |
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| 132 | 1.2385E+01 | 3.6313E-01 | 9.9991E-01 | 1.7555E-02 | 9.9987E-01 | 9.9999E-01 | 9.9973E-01 | 9.9977E-01 | 9.9977E-01 | 9.9977E-01 | 9.9977E-01 |
| 133 | 1.2653E+01 | 3.6959E-01 | 9.9991E-01 | 1.7556E-02 | 9.9987E-01 | 9.9999E-01 | 9.9973E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 |
| 134 | 1.2926E+01 | 3.7616E-01 | 9.9991E-01 | 1.7556E-02 | 9.9987E-01 | 9.9999E-01 | 9.9973E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 |
| 135 | 1.3204E+01 | 3.8287E-01 | 9.9991E-01 | 1.7556E-02 | 9.9988E-01 | 9.9999E-01 | 9.9974E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 |
| 136 | 1.3488E+01 | 3.8971E-01 | 9.9991E-01 | 1.7556E-02 | 9.9988E-01 | 9.9999E-01 | 9.9974E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 | 9.9978E-01 |
| 137 | 1.3778E+01 | 3.9669E-01 | 9.9992E-01 | 1.7556E-02 | 9.9988E-01 | 9.9999E-01 | 9.9974E-01 | 9.9979E-01 | 9.9979E-01 | 9.9979E-01 | 9.9979E-01 |
| CFF | THETA*/LF | DELTA*/LF | REX | RETHETA*F | REDELTAF | QBAR | ST | | | | |
| 1.2057E-03 | 1.7192E-02 | 8.9293E-02 | 5.1939E+04 | 2.9495E+03 | 1.5319E+04 | 1.0289E+00 | 1.0078E-02 | | | | |
| HIF | | | | | | | | | | | |
| 1.5292E+00 | | | | | | | | | | | |

Langley Research Center,
National Aeronautics and Space Administration,
Hampton, Va., February 4, 1971.

APPENDIX

LANGLEY LIBRARY SUBROUTINES

Subroutine FTLUP

Language: FORTRAN

Purpose: Computes $y = F(x)$ from a table of values using first- or second-order interpolation.
An option to give y a constant value for any x is also provided.

Use: CALL FTLUP(X, Y, M, N, VARI, VARD)

X The name of the independent variable x.

Y The name of the dependent variable $y = F(x)$.

M The order of interpolation (an integer)

$M = 0$ for y a constant. VARD(I) corresponds to VARI(I) for
 $I = 1, 2, \dots, N$. For $M = 0$ or $N \leq 1$, $y = F(VARI(1))$ for any value of x.
The program extrapolates.

$M = 1$ or 2 . First or second order if VARI is strictly increasing (not equal).

$M = -1$ or -2 . First or second order if VARI is strictly decreasing (not equal).

N The number of points in the table (an integer).

VARI The name of a one-dimensional array which contains the N values of the independent variable.

VARD The name of a one-dimensional array which contains the N values of the dependent variable.

Restrictions: All the numbers must be floating point. The values of the independent variable x in the table must be strictly increasing or strictly decreasing. The following arrays must be dimensioned by the calling program as indicated: VARI(N), VARD(N).

Accuracy: A function of the order of interpolation used.

References: (a) Nielsen, Kaj L.: Methods in Numerical Analysis. The Macmillan Co., c.1956, pp. 87-91.
(b) Milne, William Edmund: Numerical Calculus. Princeton Univ. Press, c.1949, pp. 69-73.

Storage: 430₈ locations.

Error condition: If the VARI values are not in order, the subroutine will print TABLE BELOW OUT OF ORDER FOR FTLUP AT POSITION xxx TABLE IS STORED IN LOCATION xxxxxxx (absolute). It then prints the contents of VARI and VARD, and STOPS the program.

Subroutine date: September 12, 1969.

APPENDIX – Continued

Subroutine DISCOT

Language: FORTRAN

Purpose: DISCOT performs single or double interpolation for continuous or discontinuous functions.

Given a table of some function y with two independent variables, x and z , this subroutine performs K_x th- and K_z th-order interpolation to calculate the dependent variable. In this subroutine all single-line functions are read in as two separate arrays and all multi-line functions are read in as three separate arrays; that is,

$$\begin{aligned}x_i &\quad (i = 1, 2, \dots, L) \\y_j &\quad (j = 1, 2, \dots, M) \\z_k &\quad (k = 1, 2, \dots, N)\end{aligned}$$

Use: CALL DISCOT (XA, ZA, TABX, TABY, TABZ, NC, NY, NZ, ANS)

| | |
|------|--|
| XA | The x argument |
| ZA | The z argument (may be the same name as x on single lines) |
| TABX | A one-dimensional array of x values |
| TABY | A one-dimensional array of y values |
| TABZ | A one-dimensional array of z values |
| NC | A control word that consists of a sign (+ or -) and three digits. The control word is formed as follows: <ol style="list-style-type: none">(1) If $NX = NY$, the sign is negative. If $NX \neq NY$, then NX is computed by DISCOT as $NX = NY/Nz$, and the sign is positive and may be omitted if desired.(2) A one in the hundreds position of the word indicates that no extrapolation occurs above z_{max}. With a zero in this position, extrapolation occurs when $z > z_{max}$. The zero may be omitted if desired.(3) A digit (1 to 7) in the tens position of the word indicates the order of interpolation in the x-direction.(4) A digit (1 to 7) in the units position of the word indicates the order of interpolation in the z-direction. |
| NY | The number of points in y array |
| NZ | The number of points in z array |
| ANS | The dependent variable y |

APPENDIX – Continued

The following programs will illustrate various ways to use DISCOT:

CASE I: Given $y = f(x)$
NY = 50
NX (number of points in x array) = NY
Extrapolation when $z > z_{max}$
Second-order interpolation in x-direction
No interpolation in z-direction
Control word = -020
DIMENSION TABX (50), TABY (50)
1 FORMAT (8E 9.5)
READ (5,1) TABX, TABY
READ (5,1) XA
CALL DISCOT (XA, XA, TABX, TABY, -020, 50, 0, ANS)

CASE II: Given $y = f(x,z)$
NY = 800
NZ = 10
NX = NY/NZ (computed by DISCOT)
Extrapolation when $z > z_{max}$
Linear interpolation in x-direction
Linear interpolation in z-direction
Control word = 11
DIMENSION TABX (800), TABY (800), TABZ (10)
1 FORMAT (8E 9.5)
READ (5,1) TABX, TABY, TABZ
READ (5,1) XA, ZA
CALL DISCOT (XA, ZA, TABX, TABY, TABZ, 11, 800, 10, ANS)

CASE III: Given $y = f(x,z)$
NY = 800
NZ = 10
NX = NY
Extrapolation when $z > z_{max}$
Seventh-order interpolation in x-direction
Third-order interpolation in z-direction
Control word = -73
DIMENSION TABX (800), TABY (800), TABZ (10)
1 FORMAT (8E 9.5)
READ (5,1) TABX, TABY, TABZ
READ (5,1) XA, ZA
CALL DISCOT (XA, ZA, TABX, TABY, TABZ, -73, 800, 10, ANS)

CASE IV: Same as Case III with no extrapolation above z_{max} . Control word = -173
CALL DISCOT (XA, ZA, TABX, TABY, TABZ, -173, 800, 10, ANS)

APPENDIX - Continued

Restrictions: See rule (5c) of section "Method" for restrictions on tabulating arrays and discontinuous functions. The order of interpolation in the x- and z-directions may be from 1 to 7. The following subprograms are used by DISCOT: UNS, DISSER, LAGRAN.

Method: Lagrange's interpolation formula is used in both the x- and z-directions for interpolation. This method is explained in detail in reference (a) of this subroutine. For a search in either the x- or z-direction, the following rules are observed:

- (1) If $x < x_1$, the routine chooses the following points for extrapolation:

$$x_1, x_2, \dots, x_{k+1} \text{ and } y_1, y_2, \dots, y_{k+1}$$

- (2) If $x > x_n$, the routine chooses the following points for extrapolation:

$$x_{n-k}, x_{n-k+1}, \dots, x_n \text{ and } y_{n-k}, y_{n-k+1}, \dots, y_n$$

- (3) If $x \leq x_n$, the routine chooses the following points for interpolation:

When k is odd,

$$x_{i-\frac{k+1}{2}}, x_{i-\frac{k+1}{2}+1}, \dots, x_{i-\frac{k+1}{2}+k} \text{ and } y_{i-\frac{k+1}{2}}, y_{i-\frac{k+1}{2}+1}, \dots, y_{i-\frac{k+1}{2}+k}$$

When k is even,

$$x_{i-\frac{k}{2}}, x_{i-\frac{k}{2}+1}, \dots, x_{i-\frac{k}{2}+k} \text{ and } y_{i-\frac{k}{2}}, y_{i-\frac{k}{2}+1}, \dots, y_{i-\frac{k}{2}+k}$$

- (4) If any of the subscripts in rule (3) become negative or greater than n (number of points), rules (1) and (2) apply. When discontinuous functions are tabulated, the independent variable at the point of discontinuity is repeated.

- (5) The subroutine will automatically examine the points selected before interpolation and if there is a discontinuity, the following rules apply. Let x_d and x_{d+1} be the point of discontinuity.

- (a) If $x \leq x_d$, points previously chosen are modified for interpolation as shown:

$$x_{d-k}, x_{d-k+1}, \dots, x_d \text{ and } y_{d-k}, y_{d-k+1}, \dots, y_d$$

- (b) If $x > x_d$, points previously chosen are modified for interpolation as shown:

$$x_{d+1}, x_{d+2}, \dots, x_{d+k} \text{ and } y_{d+1}, y_{d+2}, \dots, y_{d+k}$$

- (c) When tabulating discontinuous functions, there must always be $k + 1$ points above and below the discontinuity in order to get proper interpolation.

- (6) When tabulating arrays for this subroutine, both independent variables must be in ascending order.

APPENDIX – Concluded

(7) In some engineering programs with many tables, it is quite desirable to read in one array of x values that could be used for all lines of a multi-line function or different functions. Even though this situation is not always applicable, the subroutine has been written to handle it. This procedure not only saves much time in preparing tabular data, but also can save many locations previously used when every y coordinate had to have a corresponding x coordinate. Another additional feature that may be useful is the possibility of a multi-line function with no extrapolation above the top line.

Accuracy: A function of the order of interpolation used.

Reference: (a) Nielsen, Kaj L.: Methods in Numerical Analysis. The Macmillan Co., c.1956.

Storage: 555₈ locations.

Subprograms used: UNS 40₈ locations.

DISSER 110₈ locations.

LAGRAN 55₈ locations.

Subroutine date: August 1, 1968.

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